

Appendix A

Summary of Grisham's equations

A summary of the equations in Grisham's paper is given below. Note that for all equations, the buckling stress is positive.

First, the modified shear buckling allowable stress is evaluated:

$$\tau_{xy_{cr}} = \tau_{xy_{cr0}} \sqrt{1 - \frac{\sigma_x}{\sigma_{x_{cr0}}} - \frac{\sigma_y}{\sigma_{y_{cr0}}}} \quad (\text{A.1})$$

If the linear finite element solution indicates a tension-tension stress state, else for tension-compression and compression-compression the following equation is used:

$$\tau_{xy_{cr}}^2 + \frac{\tau_{xy_{cr0}}^2 (\sigma_x \sigma_{y_{cr0}} + \sigma_y \sigma_{x_{cr0}})}{|\tau_{xy}| \sigma_{x_{cr0}} \sigma_{y_{cr0}}} \tau_{xy_{cr}} - \tau_{xy_{cr0}}^2 = 0 \quad (\text{A.2})$$

Once this is done, the modified membrane buckling stresses are determined:

$$\frac{|\tau_{xy}|}{|\tau_{xy_{cr}}|} = \frac{\sigma_x}{\sigma_{x_{cr}}} = \frac{\sigma_y}{\sigma_{y_{cr}}} \quad (\text{A.3})$$

Once these have been calculated an interaction equation is used to determine if the web buckles:

$$\frac{\sigma_{x_{cr}}}{\sigma_{x_{cr0}}} + \frac{\sigma_{y_{cr}}}{\sigma_{y_{cr0}}} + \left(\frac{\tau_{xy_{cr}}}{\tau_{xy_{cr0}}} \right)^2 = 1 \quad (\text{A.4})$$

If the web buckles, the diagonal tension factor, angle and stresses in the x- and y-directions can be calculated:

$$k = \tanh\left(0.5 \log \frac{\tau_{xy}}{\tau_{xycr}}\right) \quad (\text{A.5})$$

The diagonal tension angle is calculated using the following equations and a series of successive approximations:

$$\tan^2 \alpha = \frac{\epsilon - \epsilon_f}{\epsilon - \epsilon_u} \quad (\text{A.6})$$

where:

$$\epsilon_f = \frac{H_1}{\tan \alpha} \quad (\text{A.7})$$

$$\epsilon_u = H_2 \tan \alpha \quad (\text{A.8})$$

$$\epsilon = \frac{H_3}{\sin 2\alpha} + H_4 \sin 2\alpha \quad (\text{A.9})$$

$$H_1 = \frac{-k|\tau_{xy}|}{E_f\left(\frac{A_f}{L_y t} + 0.5(1 - k)\right)} \quad (\text{A.10})$$

$$H_2 = \frac{-k|\tau_{xy}|}{E_u\left(\frac{A_u}{L_x t} + 0.5(1 - k)\right)} \quad (\text{A.11})$$

$$H_3 = \frac{2k|\tau_{xy}|}{E_w} \quad (\text{A.12})$$

$$H_4 = (1 - k + \mu - \mu k)\left(\frac{|\tau_{xy}|}{E_w}\right) \quad (\text{A.13})$$

A_f and A_u are half of the summed stiffener areas in the x- and y-directions respectively.

The diagonal tension stress to be developed in the plate following shear buckling is:

$$\sigma_{xDT} = k|\tau_{xy}| \cot \alpha \quad (\text{A.14})$$

$$\sigma_{yDT} = k|\tau_{xy}| \tan \alpha \quad (\text{A.15})$$

The shear strain of the plate in its post-buckled state is:

$$\gamma_{xy} = \left[\frac{1-k}{G_w} + \frac{k}{G_{pdt}} \right] \tau_{xy} \quad (\text{A.16})$$

where:

$$\frac{1}{G_{pdt}} = \frac{1}{2G_w(1+\mu)} \left\{ \frac{4}{\sin^2 2\alpha} + \frac{E_w L_x t \tan^2 \alpha}{E_u (A_u + 0.5 L_x t (1-k))} + \frac{E_w L_y t \cot^2 \alpha}{E_f (2A_f + 0.5 L_y t (1-k))} \right\} \quad (\text{A.17})$$

Defining:

$$\frac{1}{G_{pdt}} = \frac{1}{G_w} \theta \quad (\text{A.18})$$

then:

$$\gamma_{xy} = \left(\frac{1}{G_w} + \frac{k(\theta-1)}{G_w} \right) \tau_{xy} \quad (\text{A.19})$$

where the second term is the shear deformation of the plate in its post-buckled state.

Therefore, the set of pre-strains required to induce the diagonal tension stresses in the plate is:

$$\epsilon_{x_{DT}} = -\frac{1}{E_w} \left(\frac{E_w L_y t}{A_f E_f} + 1 \right) (\sigma_{x_{DT}} - \mu \sigma_{y_{DT}}) \quad (\text{A.20})$$

$$\epsilon_{y_{DT}} = -\frac{1}{E_w} \left(\frac{E_w L_x t}{A_u E_u} + 1 \right) (\sigma_{y_{DT}} - \mu \sigma_{x_{DT}}) \quad (\text{A.21})$$

$$\gamma_{xy_{DT}} = \frac{k(\theta-1)}{G_w} \tau_{xy} \quad (\text{A.22})$$

The compressive stresses in the buckled plate are:

$$\sigma_{x_c} = \frac{C_2 C_3 [\sigma_x - \sigma_{x_{cr}}] L + \beta_x \mu C_4 [\sigma_y - \sigma_{y_{cr}}] L}{C_1 C_2} \quad (\text{A.23})$$

$$\sigma_{y_c} = \frac{C_1 C_4 [\sigma_y - \sigma_{y_{cr}}] L + \beta_y \mu C_3 [\sigma_x - \sigma_{x_{cr}}] L}{C_1 C_2} \quad (\text{A.24})$$

where:

L is a factor to control the rate at which compressive buckling is incorporated and has a value between '0' and '1'.

$$C_1 = 1 + \frac{L_y t E_w \beta_x}{A_f E_f} \quad (\text{A.25})$$

$$C_2 = 1 + \frac{L_x t E_w \beta_y}{A_u E_u} \quad (\text{A.26})$$

$$C_3 = 1 - \beta_x \quad (\text{A.27})$$

$$C_4 = 1 - \beta_y \quad (\text{A.28})$$

and

$$\beta_x = \frac{E_{w_x}}{E_w} \quad (\text{A.29})$$

$$\beta_y = \frac{E_{w_y}}{E_w} \quad (\text{A.30})$$

The pre-strains corresponding to these changes in stress due to compression buckling are:

$$\epsilon_{x_c} = -\frac{1}{E_w} \left(1 + \frac{L_y t E_w}{A_f E_f}\right) (\sigma_x - \mu \sigma_y) |\beta_x + \beta_x \beta_y - 1| \quad (\text{A.31})$$

$$\epsilon_{y_c} = -\frac{1}{E_w} \left(1 + \frac{L_x t E_w}{A_u E_u}\right) (\sigma_y - \mu \sigma_x) |\beta_y + \beta_x \beta_y - 1| \quad (\text{A.32})$$

$$\gamma_{xy_c} = 0.0 \quad (\text{A.33})$$

The total change in strain for the first iteration is the sum of the diagonal tension strain and the compressive strain:

$$\Delta \epsilon_x = \epsilon_{x_{DT}} + \epsilon_{x_c} \quad (\text{A.34})$$

$$\Delta \epsilon_y = \epsilon_{y_{DT}} + \epsilon_{y_c} \quad (\text{A.35})$$

$$\Delta\gamma_{xy} = \frac{k(\theta - 1)}{G_w} \tau_{xy} \quad (\text{A.36})$$

These strains then become the pre-strains in the finite element model, for the next iteration. For the second and succeeding iterations, all the above equations are used. The diagonal tension stress must first be removed from the finite element stress though. The total strain for the n -th iteration is:

$$\epsilon_{x_{total}}^n = \epsilon_x^{n-1} + \Delta\epsilon_x^n \quad (\text{A.37})$$

$$\epsilon_{y_{total}}^n = \epsilon_y^{n-1} + \Delta\epsilon_y^n \quad (\text{A.38})$$

$$\gamma_{xy_{total}} = \frac{k^n(\theta^n - 1)}{G_w} \tau_{xy}^n \quad (\text{A.39})$$

Appendix B

Source code of the software developed

The Grisham Algorithm was implemented by coding the procedure in FORTRAN 77. The program consists of a main routine and numerous subprograms, each fulfilling a different task. An additional postprocessing program was also coded to extract data from the linear finite element analysis results in the ABAQUS[®] environment which could then be read by the main routine.

The code generates a flat structure of np_x by np_y panels in the x - and y - directions respectively as specified by the user. Each panel has stiffener members around its perimeter which may have totally different cross-sectional areas for each member. Once the number of elements per flange and the elements per upright are chosen, they remain the same for all flanges and all uprights.

The program also makes provision for all possible configurations of boundary conditions for the web buckling critical values (all sides simply supported; all sides fixed; 2 horizontal sides fixed and 2 vertical sides simply supported; 2 horizontal sides simply supported and 2 vertical sides fixed).

The web can be modelled using either shell or membrane elements. The flanges and uprights can be modelled using beam or truss elements. Elements can be first or second order. Only buckling of the web is considered, buckling of the uprights is not taken into account.

```

C
C   VARIABLE DEFINITIONS: (DOUBLE PRECISION)
C
C   ALFA      Diagonal tension angle;  positive when measured
C             from the local x- axis using the right hand rule
C   ALFA11    Expansion coefficient in the x- direction
C   ALFA22    Expansion coefficient in the y- direction
C   ALFA12    Expansion coefficient - shear
C   AX        Half the summed stiffener areas in the x- direction
C             of each panel
C   AY        Half the summed stiffener areas in the y- direction
C             of each panel
C-----
C   B          Shortest side of plate (minimum of LX or LY)
C   BETAX/Y    Parameter used to indicate whether pre- or post-buckling
C             modulus of elasticity is applicable in the x- or y-
C             direction of the plate/sheet
C   BBETAX/Y  Coefficients used in incremental change in strain equations
C             (second and succeeding iterations) to assist convergence of
C             previously compressively buckled plates that have become
C             unbuckled
C   BNF1       Beam nodal force - component 1
C   BNF2       Beam nodal force - component 2
C   BNF3       Beam nodal force - component 3
C   BUCK       The interaction equation used for bi-axial compression
C             and shear buckling
C-----
C   C1-C4     Variables used in calculating the compressive stresses in
C             a plate
C   CONV1X     Convergence requirement 1
C   CONV1Y     Convergence requirement 1
C   CONV2X     Convergence requirement 2
C   CONV2Y     Convergence requirement 2
C-----
C   DFX12     Difference in total nodal force between points 1 and 2
C             of the panel in the x-direction (normal to side)
C   DFX23     Difference in total nodal force between points 2 and 3
C             of the panel in the x-direction (normal to side)
C   DFX34     Difference in total nodal force between points 3 and 4
C             of the panel in the x-direction (normal to side)
C   DFX41     Difference in total nodal force between points 4 and 1
C             of the panel in the x-direction (normal to side)
C   DFY12     Difference in total nodal force between points 1 and 2
C             of the panel, in the y-direction (// to side)
C   DFY23     Difference in total nodal force between points 2 and 3

```

```

C          of the panel, in the y-direction (// to side)
C    DFY34  Difference in total nodal force between points 3 and 4
C          of the panel, in the y-direction (// to side)
C    DFY41  Difference in total nodal force between points 4 and 1
C          of the panel, in the y-direction (// to side)
C-----
C    ENODE  Eccentricity of uprights !! (not flanges) - node to node
C    ECC    Eccentricity of uprights - centroid to centroid
C    EP     Modulus of Elasticity of plate
C    EQN1X  Equation used in first test for BETAs
C    EQN1Y  Equation used in first test for BETAs
C    EQN2X  Equation used in second test for BETAs
C    EQN2Y  Equation used in second test for BETAs
C    EQN3X  Equation used in third test for BETAs
C    EQN3Y  Equation used in third test for BETAs
C    ERR    % error made while calculating successive approximations of the
C          diagonal tension angle (ALFA)
C    ES     Minimum % error allowed in calculating root of
C          equation (subroutine)
C    EXB    Modulus of Elasticity of beam in x- direction
C    EXC    Strain in the x-direction due to compressive loading
C          only
C    EXDT   Strain in the x- direction due to diagonal tension
C    EXP    Effective Modulus of Elasticity of buckled plate in x-
C          direction
C    EXT    Total strain in the x- direction [compressive loading +
C          diagonal tension]
C    EXYDT  Shear strain due to diagonal tension
C    EXYT   Total shear strain
C    EYB    Modulus of Elasticity of beam in y-direction
C    EYC    Strain in the y- direction due to compressive loading
C          only
C    EYDT   Strain in the y- direction due to diagonal tension
C    EYP    Effective Modulus of Elasticity of buckled plate in y-
C          direction
C    EYT    Total strain in the y- direction[compressive loading +
C          diagonal tension]
C-----
C    FLA    Flange T-section dimension
C    FLB    Flange T-section dimension
C    FLT1   Flange T-section dimension
C    FLT2   Flange T-section dimension
C-----
C    GP     Shear modulus of plate
C-----

```



```

C      H1-H4      Variables used to calculate the diagonal tension angle in
C                  a plate
C-----
C      IXX       Moment of Inertia of upright about an axis through its own
C                  centroid and parallel to the web plane
C-----
C      K         Diagonal tension factor
C      KS       Shear buckling coefficient for a flat plate
C      KX       Buckling coefficient in x-direction for a flat plate
C      KY       Buckling coefficient in y-direction for a flat plate
C-----
C      L         Factor between 0.0 and 1.0 to control the rate at which
C                  buckling is incorporated in the solution
C      L1       if LX >= LY then L1 = LY and L2 = LX
C      L2       if LY > LX then L1 = LX and L2 = Ly
C      L12      Plate side length; between corner nodes 1 and 2
C      L23      Plate side length; between corner nodes 2 and 3
C      L34      Plate side length; between corner nodes 3 and 4
C      L41      Plate side length; between corner nodes 4 and 1
C      LPNX     Length between nodes in the x-direction
C      LPNY     Length between nodes in the y-direction
C      LTOTX    Total length of model in x-direction
C      LTOTY    Total length of model in y-direction
C      LX       Length of plate in x-direction
C      LY       Length of plate in y-direction
C-----
C      NCX      Nodal x-coordinate generated within program
C      NCY      Nodal y-coordinate generated within program
C      NCZ      Nodal z-coordinate generated within program
C      NF1T     Total force at node - component 1 read from ABAQUS output
C      NF2T     Total force at node - component 2 read from ABAQUS output
C      NF3T     Total force at node - component 3 read from ABAQUS output
C-----
C      POISS    Poisson's ratio
C-----
C      RHO      Radius of gyration of the upright area
C-----
C      SNF1     Shell nodal force - component 1
C      SNF2     Shell nodal force - component 2
C      SNF3     Shell nodal force - component 3
C      SX       Average panel normal stress in the x-direction
C      SX12     Normal stress to plate edge between nodes 1 and 2
C      SX23     Normal stress to plate edge between nodes 2 and 3
C      SX34     Normal stress to plate edge between nodes 3 and 4
C      SX41     Normal stress to plate edge between nodes 4 and 1

```

```

C     SXC      Total compressive stress acting on plate in x-direction
C     SXCR     Modified buckling allowable stress in x- direction
C     SXCRO    Critical buckling stress in x-direction (geometry only)
C     SXDT     Diagonal tension stress in plate following buckling
C     SXEFF    Effective (resultant) tensile/compressive stress applied to
C             panel/sheet calculated from panel/sheet nodal
C             forces [x-direction]
C     SXY      Average panel shear stress
C     SXY12    Shear stress alongside plate edge between nodes 1 and 2
C     SXY23    Shear stress alongside plate edge between nodes 2 and 3
C     SXY34    Shear stress alongside plate edge between nodes 3 and 4
C     SXY41    Shear stress alongside plate edge between nodes 4 and 1
C     SXYEFF   Effective (resultant) shear stress applied to panel/sheet
C             calculated form panel/sheet nodal forces [same value on each
C             of the 4 sides]
C     SXYCR    Modified buckling shear allowable stress
C     SXYCRO   Critical shear buckling stress (geometry only)
C     SY       Average panel normal stress in the y-direction
C     SYC      Total compressive stress acting on plate in y-direction
C     SYCR     Modified buckling allowable stress in y- direction
C     SYCRO    Critical buckling stress in y-direction (geometry only)
C     SYDT     Diagonal tension stress in plate following buckling
C     SYEFF    Effective (resultant) tensile/compressive stress applied to
C             panel/sheet calculated from panel/sheet nodal
C             forces [x-direction]
C-----
C     T        Plate/sheet thickness
C     TDB      Buckling equation rounded to four decimal places value
C     THETA    Diagonal tension angle
C-----
C     UPRA     Angle cross-section dimension
C     UPRT     Angle cross-section thickness dimension
C     UPRY     Angle cross-section dimension
C-----
C     XALFA    Guestimate of diagonal tension angle in method of successive
C             approximations to determine ALFA
C     XN       The root of the equation (subroutine)
C     XR1      Initial guess of root of equation (subroutine)
C     XR2      Initial guess of root of equation (subroutine)
C-----
C     YCENT    Centroid position of angled upright
C-----
C
C
C     VARIABLE DEFINITIONS: (INTEGERS)

```

```

C
C   BENF      Beam element node force array used with node forces
C   BNNNF     Beam node number for node force data
C   BTOT     Total number of panels that are not buckled at the end
C            of the analysis
C-----
C   C20       Constant that = 2 when second order elements are used
C-----
C   ELBH     Horizontal beam element set
C   ELBHN    Horizontal beam element number
C   ELBV     Vertical beam element set
C   ELBVN    Vertical beam element number
C   ELS      Shell element set
C   ELSN     Shell element number
C-----
C   FN       File number for each panel
C   FN       File number for each panel
C-----
C   IC1,2..  Counters
C   ICB      Beam element counter
C   ICBH     Horizontal beam element counter
C   ICBV     Vertical beam element counter
C   ICS      Shell element counter
C   IM       Maximum number of iterations allowed to determine root of
C            equation (subroutine)
C   ITN      Iteration number
C-----
C   K1,K2..  Constants
C-----
C   NALL     All the nodes in the linear FEA model
C   NBEPP    Total number of beam elements associated with each panel
C   NEX      Number of elements per panel/sheet along the x-axis
C   NEY      Number of elements per panel/sheet along the y-axis
C   NFLS     Number of flange element sets
C   NI       Same as ITER
C   NINCY    Node number increment per row along the y-axis
C   NIPPX    Node increment per panel/sheet in the x-direction
C   NIPPY    Node increment per panel/sheet in the y-direction
C   NNXMAX   Maximum node number value in the x-direction
C   NNYMAX   Maximum node number value in the y-direction
C   NONX     Total number of nodes along the x-axis
C   NONY     Total number of nodes along the y-axis
C   NP       Number of panels in structure
C   NPX      Number of panels/sheets along the x-axis
C   NPY      Number of panels/sheets along the y-axis\

```

```

C      NRBNF      Number of records in the beam element node forces file
C      NSEPP      Total number of shell elements associated with each panel
C      NTOTAL     Total number of nodes generated
C-----
C      PCNN       Panel corner node number
C      PNLBE      Panel beam elements (panel number,panel beam element
C                number - random)
C      PNLNN      Panel corner node numbers [4] (panel number,panel corner node
C                number - clockwise; must be clockwise and start at 0,0,0
C                so that stresses in each direction can be calculated
C                correctly)
C      PNLSE      Panel shell elements (panel number,panel shell element
C                number - random)
C-----
C      SHENF      Shell element node force array used with node forces
C      SHNNNF     Shell node number for node force data
C-----
C      YINC       Node number increment per row along the y-axis;
C                = relevant NINCY value
C-----
C
C*****
C                VARIABLE DEFINITIONS:  (CHARACTERS)
C
C      COMMA      A comma
C      UPRE       Upright eccentricity presence (=NOE or ECC)
C
C*****
C*****
C
C                MAIN PROGRAM
C
C*****
C*****
C
C      DOUBLE PRECISION ALFA,
C      +AX,AY,
C      +B,BETAX(301),BETAY(301),BBETAX,BBETAY,
C      +EP,ERR,EXB,EXP,EYB,EYP,ECC(301),ENODE(301),
C      +EQN1X,EQN1Y,EQN2X,EQN2Y,EQN3X,EQN3Y,
C      +FLA(301),FLB(301),FLT1(301),FLT2(301),
C      +GP,
C      +H1,H2,H3,H4,
C      +IXX(301),

```

```

+K,KS,KX,KY,
+L,L1,L2,LPNX,LPNY,LTOTX,LTOTY,LX,LY,
+POISS,PI,
+RHO(301),
+SXCRO,SYCRO,SXYCRO,
+T(301),THETA,TDB,
+UPRA(301),UPRT(301),UPRY(301),
+XALFA,
+YCENT(301),
+SXAV,SYAV,SXYAV,SX,SY,SXY,BB,SXYCR1,SXYCR2,SXYCR,
+SXCR,SYCR,SXCRI(301),SYCRI(301),SXM0D,SYM0D,
+BUCK,EPSX,EPsy,EPs,
+SXDT(301),SYDT(301),EXDT,EYDT,EXYDT,C1,C2,C3,C4,
+SXC,SYC,EXC,EYC,EXT(301),EYT(301),EXYT,
+SXPS(301),SYPS(301),SXYPS(301),
+SXDTI(301),SYDTI(301),SXDTII(301),SYDTII(301),
+NCX(1000),NCY(1000),NCZ(1000),
+S11(1000),S22(1000),S12(1000),
+SXAVE(301),SYAVE(301),SXYAVE(301),
+SXAVP(301),SYAVP(301),SXYAVP(301),
+SXTE(1000),SYTE(1000),SXYTE(1000),
+SXTP(1000),SYTP(1000),SXYTP(1000),
+K1X,K1Y,K1XP,K1YP,K2X,K2Y,K2XP,K2YP,K1XN(301),K1YN(301),
+K1XNN(301),K1YNN(301),K2XN(301),K2YN(301),K1XPP,K1YPP,
+K2XPP,K2YPP,K2XNN(301),K2YNN(301),K2XPPP,K2YPPP,
+K1XVAL(301),K1YVAL(301),
+K1XNC(301),K1YNC(301),PVAL,
+UPRRAD(301),FLRAD(301),AFL(301),AUPR(301),AUPRE(301)

```

C

```

INTEGER BHTOT,BVTOT,C20,
+CONA(301),CONB(301),CONC(301),COND(301),
+BCONA(301),BCONB(301),
+CTOT,BCTOT,
+FN,FN2,
+ITN,
+NBEPP,NP,NFLS,NRBNF,NINCY,
+NSEPP,NUPRS,
+SEC,
+TSTART,TSTOP,
+YINC,
+ELE(1000),
+ELBH(1000,1000),ELBHN(1000),ELBV(1000,1000),ELBVN(1000),
+ElsN(1000),
+IPT(1000),SPT(1000),
+NALL(1000),

```

```

+PCNN(1000),ELS(1000,1000),
+PNLNN(1000,6),PNLSE(1000,20),PNLBE(1000,20),
+K1XID(301),K1YID(301),
+BPL(301),BTOT,BPID(301),
+CONAT,CONBT,CONCT,CONDT,
+IBCMAX,IFAC
C
  CHARACTER*(1)COMMA
  CHARACTER*(3)UPRE
C
  OPEN(10,FILE='buk1.dat')
  OPEN(21,FILE='conv1.dat')
  OPEN(12,FILE='inputdata.dat')
  OPEN(13,FILE='conv2.dat')
  OPEN(31,FILE='pnl1-x.dat')
  OPEN(32,FILE='pnl2-x.dat')
  OPEN(33,FILE='pnl3-x.dat')
  OPEN(34,FILE='pnl4-x.dat')
  OPEN(35,FILE='pnl5-x.dat')
  OPEN(36,FILE='pnl6-x.dat')
  OPEN(41,FILE='pnl1-y.dat')
  OPEN(42,FILE='pnl2-y.dat')
  OPEN(43,FILE='pnl3-y.dat')
  OPEN(44,FILE='pnl4-y.dat')
  OPEN(45,FILE='pnl5-y.dat')
  OPEN(46,FILE='pnl6-y.dat')
C
C ++++++
C
C   INPUT DATA:
C ++++++
C
C
  NPX=6
  NPY=1
  NEX=3
  NEY=3
  LX=0.254
  LY=0.7254
  NP=NPX*NPY
C
  DO 63 I=1,NP+1
    UPRA(I)=0.0254
    UPRT(I)=0.003175

```

```

63  CONTINUE
C
    UPRE='ECC'
C
    DO 64 J=1,NP
        T(J)=0.000635
        FLRAD(J)=0.00881
        FLRAD(NP+J)=0.01177
64  CONTINUE
C
    EXB=71.0E9
    EYB=71.0E9
    EP=72.4E9
    GP=26.92E9
    POISS=0.3
    C20=2
C
    COMMA=', '
    PVAL=0.0
C
    IBCMAX=60
    PI=3.14159265359
C
C ++++++
C
C  PROGRAM STARTS
C
C ++++++
C
    CALL TIME(SEC)
    TSTART=SEC
    WRITE(10,10)TSTART
10  FORMAT('TSTART = ',I20)
C
C          geometric calculations for structure
C
    LTOTX=LX*NPX
    LTOTY=LY*NPY
    NNXMAX=C20*NPX*NEX+1
    NONX=NNXMAX
    NONY=C20*NPY*NEY+1
    NIPPX=C20*NEX
    NBEPP=2*NEX+2*NEY
    NSEPP=NEX*NEY
    LPNX=LTOTX/(NONX-1)

```

```

LPNY=LTOTY/(NONY-1)
NINCY=NNXMAX
NNYMAX=C20*NPY*NEY+1
NTOTAL=NNXMAX*NNYMAX
NIPPY=C20*NEY*NINCY
C
DO 67 I=1,NPX*NPY+NPX
  AFL(I)=PI*FLRAD(I)**2
67 CONTINUE
C
C           Geometric properties of uprights calculated
C
DO 14 I=1,NPX+1
  AUPR(I)=(UPRA(I)-UPRT(I))*UPRT(I)+UPRA(I)*UPRT(I)
  YCENT(I)=((UPRA(I)-UPRT(I))*UPRT(I)*UPRT(I)/2.0+
+           UPRA(I)*UPRT(I)*UPRA(I)/2.0)/((UPRA(I)-
+           UPRT(I))*UPRT(I)+UPRA(I)*UPRT(I))
  UPRY(I)=UPRA(I)-YCENT(I)
  IXX(I)=(1.0/3.0)*(UPRT(I)*UPRY(I)**3+UPRA(I)*
+           (UPRA(I)-UPRY(I))**3-(UPRA(I)-UPRT(I))*
+           (UPRA(I)-UPRY(I)-UPRT(I))**3)
  RHO(I)=SQRT(IXX(I)/AUPR(I))
  IF(I.EQ.1)THEN
    ECC(I)=YCENT(I)+T(I)/2.0
    ENODE(I)=-T(I)/2.0+UPRT(I)/2.0
  ELSE IF(I.EQ.NPX+1)THEN
    ECC(I)=YCENT(I)+T(I-1)/2.0
    ENODE(I)=-T(I-1)/2.0+UPRT(I)/2.0
  ELSE
    ECC(I)=YCENT(I)+(T(I-1)+T(I))/4.0
    ENODE(I)=-((T(I-1)+T(I))/4.0+UPRT(I)/2.0)
  ENDIF
  AUPRE(I)=AUPR(I)/(1+(ECC(I)/RHO(I))**2)
  WRITE(10,15)I,AUPR(I),YCENT(I),IXX(I),RHO(I),
+ECC(I),AUPRE(I),ENODE(I)
15  FORMAT(/,'  Upright No',I3,/,
+'AUPR = ',G15.5,/,
+'YCENT = ',G15.5,'      IXX = ',G20.5,/,
+'RHO = ',G15.5,'      ECC = ',G15.5,/,
+'AUPRE = ',G15.5,'      ENODE = ',G15.5)
14  CONTINUE
C
CALL FNODES(NPX,NPY,NEX,NEY,C20,LTOTX,LTOTY,UPRE,
+ENODE,NALL,NCX,NCY,NCZ,NTOT,NTFLAT)
PRINT *, ' Finished subroutine fnodes !@!!!!!'

```



```

C
  CALL ELEMENTS(NPX,NPY,NEX,NEY,C20,UPRE,NALL,ELS,ELBH,
+ELBV,ELBVN,ELBHN,ELSN,PNLSE,ICS,ICBH,ICBV,NTFLAT)
  PRINT *,' Finished subroutine ele !!!!'

C
C      Iterative loop to determine best value for BETAX and BETAY
C      for each panel to satisfy convergence !
C
  DO 650 I=1,NP
    BETAX(I)=0.85
    BETAY(I)=0.85
650  CONTINUE

C
  DO 3500 IBC=1,IBCMAX

C
  PRINT *,' IBC = ',IBC
  WRITE(10,651)IBC
  WRITE(13,651)IBC
651  FORMAT(/,' %%%%%%%%%%% Loop',I3,' for the BETAs',
+ ' convergence iterations ! %%%%%%%%%%%',/)

C
  DO 655 I=1,NP
    BPL(I)=0
    BPID(I)=0
    SXCRI(I)=0.0
    SYCRI(I)=0.0
    SXDTI(I)=0.0
    SYDTI(I)=0.0
    SXDTII(I)=0.0
    SYDTII(I)=0.0
    K1XN(I)=0.0
    K1YN(I)=0.0
    K1XNN(I)=0.0
    K1YNN(I)=0.0
    K2XN(I)=0.0
    K2YN(I)=0.0
    K2XNN(I)=0.0
    K2YNN(I)=0.0
    K1XVAL(I)=0
    K1YVAL(I)=0
    CONA(I)=0
    CONB(I)=0
    CONC(I)=0
    COND(I)=0
655  CONTINUE

```

```

C
C ***** ITERATION LOOP STARTS HERE *****
C
      DO 658 FN=31,36,1
      WRITE(FN,657)
657  FORMAT(/,'IBC ITN K1X=SXC/SXCR',4X,'SXC',3X,'SXCR=SXYCR/SXY',
+4X,'SXYCR',8X,'SXY',6X,'SX-SXCR',7X,'SY-SYCR',8X,'SX',8X,'SXAV',
+6X,'SXDT(I)')
658  CONTINUE
      DO 661 FN2=41,46,1
      WRITE(FN2,662)
662  FORMAT(/,'IBC ITN K1Y=SYC/SYCR',4X,'SYC',3X,'SYCR=SXYCR/SXY',
+4X,'SXYCR',8X,'SXY',6X,'SY-SYCR',7X,'SX-SXCR',8X,'SY',8X,'SYAV',
+6X,'SYDT(I)')
661  CONTINUE
C
      DO 3000 ITN=1,20
C
      WRITE(10,692)ITN
      WRITE(21,692)ITN
692  FORMAT(/,'##### Iteration ',I2,
+' #####',/)
C
      DO 690 I=1,NP
      SXTP(I)=0.0
      SYTP(I)=0.0
      SXYTP(I)=0.0
      SXAVP(I)=0.0
      SYAVP(I)=0.0
      SXYAVP(I)=0.0
690  CONTINUE
C
      DO 691 J=1,ICS
      SXTE(J)=0.0
      SYTE(J)=0.0
      SXYTE(J)=0.0
      SXAVE(J)=0.0
      SYAVE(J)=0.0
      SXYAVE(J)=0.0
691  CONTINUE
C
      CALL FEMINP(NPX,NPY,NEX,NEY,C20,ICS,ICBH,ICBV,
+NTFLAT,NTOT,PNLSE,ITN,UPRE,T,SXPS,SYPS,SXYPS,NALL,
+NCX,NCY,NCZ,ELS,ELSN,ELBHN,ELBH,ELBVN,ELBV)
C

```

```

        CALL SYSTEM("abq58 job=femmodel interactive")
C
        CALL SYSTEM("strdata.x")
C
        read in the stresses for the panel elements
C
        NRBNF=0
        WRITE(12,1082)
1082  FORMAT(/,2X'ELE',3X,'INTGR PT',9X,'SECT PT',11X,'S11',11X,'S22',
+3X,'S12')
        OPEN(4,FILE='panelstress.txt')
        DO 1080 I=1,10000000
            READ(4,*,END=1081)ELE(I),IPT(I),SPT(I),S11(I),S22(I),S12(I)
            WRITE(12,*)ELE(I),IPT(I),SPT(I),S11(I),S22(I),S12(I)
1080  CONTINUE
1081  NRBNF=I-1
        PRINT *,'NRBNF =',NRBNF
        CLOSE(4)
C
        DO 1090 I=1,ICS
            ICES=0
            WRITE(10,1092)ELSN(I)
1092  FORMAT(/,'Element Number:',I4,/, '-----',
+/,8X,'Integr pt',
+ 6X,'S11',17X,'S22',17X,'S12')
            DO 1095 J=1,NRBNF
                IF(ELSN(I).EQ.ELE(J))THEN
                    SXTE(I)=SXTE(I)+S11(J)
                    SYTE(I)=SYTE(I)+S22(J)
                    SXYTE(I)=SXYTE(I)+S12(J)
                    ICES=ICES+1
                    WRITE(10,1096)IPT(J),S11(J),S22(J),S12(J)
1096  FORMAT(10X,I4,6X,F18.3,2X,F18.3,2X,F18.3)
                ENDIF
1095  CONTINUE
            WRITE(10,1099)ICES,SXTE(I),SYTE(I),SXYTE(I)
1099  FORMAT(/,'TOTAL:',4X,I4,10X,F18.3,2X,F18.3,2X,F18.3)
            SXAVE(I)=SXTE(I)/ICES
            SYAVE(I)=SYTE(I)/ICES
            SXYAVE(I)=SXYTE(I)/ICES
            WRITE(10,1098)SXAVE(I),SYAVE(I),SXYAVE(I)
1098  FORMAT(/,'AVERAGE:',12X,F18.3,2X,F18.3,2X,F18.3,/)
1090  CONTINUE
C
        DO 1120 I=1,NP

```

```

        WRITE(10,1122)I
1122  FORMAT(/,'Panel Number: ',I3,/, '-----',/,
+ 9X,'SXAVE',16X,'SYAVE',13X,
+ 'SXYAVE')
        DO 1130 J=1,NSEPP
        DO 1131 K=1,ICS
            IF(PNLSE(I,J).EQ.ELSN(K))THEN
                SXTP(I)=SXTP(I)+SXAVE(K)
                SYTP(I)=SYTP(I)+SYAVE(K)
                SXYTP(I)=SXYTP(I)+SXYAVE(K)
                WRITE(10,1140)SXAVE(K),SYAVE(K),SXYAVE(K)
1140  FORMAT(20X,F18.3,2X,F18.3,2X,F18.3)
            ENDIF
1131  CONTINUE
1130  CONTINUE
        WRITE(10,1125)SXTP(I),SYTP(I),SXYTP(I)
1125  FORMAT(/,'TOTAL:',18X,F18.3,2X,F18.3,2X,F18.3)
        SXAVP(I)=SXTP(I)/NSEPP
        SYAVP(I)=SYTP(I)/NSEPP
        SXYAVP(I)=SXYTP(I)/NSEPP
        WRITE(10,1128)SXAVP(I),SYAVP(I),SXYAVP(I)
1128  FORMAT(/,'AVERAGE:',12X,F18.3,2X,F18.3,2X,F18.3,/)
1120  CONTINUE
C
C          calculate average panel stress
C
        NPC=0
        BTOT=0
        CTOT=0
        CONAT=0
        CONBT=0
        CONCT=0
        CONDT=0
        DO 1163 IJ=1,NP
            K1XNC(IJ)=0
            K1YNC(IJ)=0
            K1XID(IJ)=0
            K1YID(IJ)=0
1163  CONTINUE
        WRITE(13,1166)IBC,ITN
1166  FORMAT('IBC = ',I4,/, 'ITN = ',I4)
C
C          Individual panel loop !
C
        XFAC1=0.0

```

```

        IFAC=0
        DO 3002 I=1,NP
        WRITE(10,1161)I,ITN
1161  FORMAT(/,1X,'***** Analyzing Panel No',I3,
        +' / Iteration No',I3,' *****',/)
C
        SXAV=SXAVP(I)
        SYAV=SYAVP(I)
        SXYAV=SXYAVP(I)
        WRITE(10,1162)SXAV,SYAV,SXYAV
1162  FORMAT('SXAV = ',F18.3,'    SYAV = ',F18.3,'    SXYAV = ',F18.3)
C
        WRITE(*,1031)AFL(I),AFL(I+NPX)
        WRITE(10,1031)AFL(I),AFL(I+NPX)
1031  FORMAT('AFL(I) = ',F15.10,'    AFL(I+NPX) = ',F15.10)
        AX=0.5*(AFL(I)+AFL(I+NPX))
        IF(UPRE.EQ.'ECC')THEN
            AY=AUPRE(I)
        ELSE
            AY=0.5*(AUPR(I+IFAC)+AUPR(I+1+IFAC))
        ENDIF
        IFAC=INT(I/NPX)
        WRITE(10,1021)AX,AY,IFAC
1021  FORMAT('AX = ',F15.10,'    AY = ',F15.10,/, 'IFAC = ',I4)
C
C      Buckling coefficients
C
        KX=1.985*((LY**2)/(LX**2))+0.941*(LY/LX)+6.31
        KY=1.985*((LX**2)/(LY**2))+0.941*(LX/LY)+6.31
        IF(LX.GE.LY)THEN
            according to ILENGTH requirements !
            L1=LY
            L2=LX
        ELSE
            L1=LX
            L2=LY
        ENDIF
        KS=5.21*((L1**2)/(L2**2))+0.14*(L1/L2)+8.05
C
        WRITE(10,FMT=1020)KX,KY,KS,L1,L2
1020  FORMAT('KX = ',F15.9,/, 'KY = ',F15.9,/, 'KS = ',
        +F15.9,/, 'L1 = ',F15.9,/, 'L2 = ',F15.9)
        IF(LX.GE.LY)THEN
            B=LY
        ELSE

```

```

        B=LX
        ENDIF
        WRITE(10,FMT=1042)B
1042  FORMAT('B = ',F15.9)
C
        SXCRO=KX*EP*((T(I)/LY)**2)
        SYCRO=KY*EP*((T(I)/LX)**2)
        SXYCRO=KS*EP*((T(I)/B)**2)
        WRITE(10,FMT=1045)SXCRO,SYCRO,SXYCRO
1045  FORMAT('SXCRO = ',F15.3,3X,'SYCRO = ',F15.3,3X,'SXYCRO = ',F15.3)
C
        L=1.0
        IF(ITN.EQ.1)THEN
C
        SX=-1.0*SXAV
        SY=-1.0*SYAV
        SXY=SXYAV
        WRITE(10,1200)SX,SY,SXY
1200  FORMAT('SX = ',F18.3,' SY = ',F18.3,' SXY = ',F18.3)
C
        IF(SX.LT.0.0.AND.SY.LT.0.0)THEN
        SXYCR=SXYCRO*((1-SX/SXCRO-SY/SYCRO)**0.5)
        ELSE
        BB=((SXYCRO**2)*(SX*SYCRO+SY*SXCRO))/(ABS(SXY)*SXCRO*SYCRO)
        SXYCR1=(-BB+(BB**2+4*SXYCRO**2)**0.5)/2.0
        SXYCR2=(-BB-(BB**2+4*SXYCRO**2)**0.5)/2.0
        WRITE(10,1210)BB,SXYCR1,SXYCR2
1210  FORMAT('BB = ',F18.3,/, 'SXYCR1 = ',F18.3,/,
+ 'SXYCR2 = ',F18.3)
        IF(ABS(SXYCR1).GE.ABS(SXYCR2))THEN
        SXYCR=SXYCR1
        ELSE
        SXYCR=SXYCR2
        ENDIF
        ENDIF
        SXCR=SX*(ABS(SXYCR)/ABS(SXY))
        SYCR=SY*(ABS(SXYCR)/ABS(SXY))
        WRITE(10,1220)SXYCR,SXCR,SYCR
1220  FORMAT('SXYCR= ',F18.3,' SXCR= ',F18.3,' SYCR= ',F18.3)
C
        BUCKT1=SXCR/SXCRO
        BUCKT2=SYCR/SYCRO
        BUCKT3=(SXYCR/SXYCRO)**2
        WRITE(10,1215)BUCKT1,BUCKT2,BUCKT3
1215  FORMAT('BUCKT1 = ',F20.10,/, 'BUCKT2 = ',F20.10,/,

```

```

+ 'BUCKT3 = ',F20.10)
BUCK=SXCR/SXCRO+SYCR/SYCRO+(SXYCR/SXYCRO)**2
TDB=NINT(BUCK*10000.0)/10000.0
WRITE(10,1222)BUCK,TDB
1222 FORMAT('Buckling Equation = ',F20.10,/, 'TDB = ',F20.4)
C
IF(TDB.LT.1.0)THEN
  BPL(I)=BPL(I)+1
WRITE(10,1224)I
1224 FORMAT(' Panel no',I2,' does not buckle !',/)
GOTO 3002
ENDIF
C
K=TANH(0.5*LOG10(ABS(SXY)/ABS(SXYCR)))
C
H1=(-K*ABS(SXY))/(EXB*(2.0*AX/LY/T(I)+0.5*(1-K)))
H2=(-K*ABS(SXY))/(EYB*(AY/LX/T(I)+0.5*(1-K)))
H3=ABS(SXY)*2.0*K/EP
H4=(1-K+POISS-POISS*K)*(ABS(SXY)/EP)
WRITE(10,1230)K,H1,H2,H3,H4
1230 FORMAT('K = ',F7.5,/,
+'H1 = ',F15.11,' H2 = ',F15.11,' H3 = ',F15.11,
+' H4 = ',F15.11)
C
XALFA=PI/4.0
1240 WRITE(10,1245)XALFA
1245 FORMAT('XALFA = ',F10.7)
EPSX=H1/TAN(XALFA)
EPSY=H2*TAN(XALFA)
EPS=H3/SIN(2*XALFA)+H4*SIN(2*XALFA)
WRITE(10,1250)EPSX,EPSY,EPS
1250 FORMAT('EPSX = ',F10.8,' EPSY = ',F10.8,' EPS = ',F10.8)
C
IF((EPS-EPSX).LT.0.0.AND.(EPS-EPSY).GT.0.0)THEN
  PRINT *,' Diagonal tension angle cannot be calculated !'
  GOTO 3001
ELSE IF((EPS-EPSX).GT.0.0.AND.(EPS-EPSY).LT.0.0)THEN
  PRINT *,' Diagonal tension angle cannot be calculated !'
  GOTO 3001
ELSE IF((EPS-EPSY).EQ.0.0)THEN
  PRINT *,' Diagonal tension angle cannot be calculated !'
  GOTO 3001
ENDIF
ALFA=ATAN(((EPS-EPSX)/(EPS-EPSY))*0.5)
ERR=(ABS(ALFA-XALFA))*100.0

```

```

WRITE(10,1255)ALFA,ERR
1255 FORMAT('ALFA = ',F10.7,' % ERR = ',F6.3)
IF(ERR.GE.0.1)THEN
  XALFA=ALFA
  GOTO 1240
ENDIF
WRITE(10,1256)I,ALFA*180.0/3.141596
1256 FORMAT(/,' Diagonal Tension Angle for Panel',I3,' = ',F6.3,
+' degrees !',/)
C
SXDT(I)=K*ABS(SXY)/TAN(ALFA)
SYDT(I)=K*ABS(SXY)*TAN(ALFA)
WRITE(10,1260)SXDT(I),SYDT(I)
1260 FORMAT('SXDT(I) = ',F18.3,'SYDT(I) = ',F18.3)
C
EXDT=(-1.0/EP)*(EP*LY*T(I)/2.0/AX/EXB+1)*(SXDT(I)-POISS*SYDT(I))
EYDT=(-1.0/EP)*(EP*LX*T(I)/AY/EYB+1)*(SYDT(I)-POISS*SXDT(I))
WRITE(10,1265)EXDT,EYDT
1265 FORMAT('EXDT = ',F13.10,' EYDT = ',F13.10)
C
THETA=(1.0/(2*(1.0+POISS)))*(4.0/(SIN(2*ALFA))**2+
+(EP*LX*T(I)*(TAN(ALFA))**2)/(EYB*(AY+(0.5*LX*T(I))*(1-K)))+
+(EP*LY*T(I)/(TAN(ALFA))**2)/(EXB*(2.0*AX+(0.5*LY*T(I))*(1-K))))
EXYDT=K*(THETA-1)*SXY/GP
C
C1=1+LY*T(I)*EP*BETAX(I)/2.0/AX/EXB
C2=1+LX*T(I)*EP*BETAY(I)/AY/EYB
C3=1-BETAX(I)
C4=1-BETAY(I)
WRITE(10,1270)THETA,EXYDT,BETAX(I),BETAY(I),C1,C2,C3,C4
1270 FORMAT('THETA = ',F10.6,' EXYDT = ',F13.10,/,
+'BETAX(I) = ',F10.6,' BETAY(I) = ',F10.6,/,
+'C1 = ',F10.6,' C2 = ',F10.6,' C3 = ',F10.6,' C4 = ',F10.6)
C
SXC=(C2*C3*(SX-SXCR)*L+BETAX(I)*POISS*C4*(SY-SYCR)*L)/C1/C2
SYC=(C1*C4*(SY-SYCR)*L+BETAY(I)*POISS*C3*(SX-SXCR)*L)/C1/C2
WRITE(10,1275)SXC,SYC
1275 FORMAT('SXC = ',F18.3,' SYC = ',F18.3)
C
EXC=(-1.0/EP)*(1+LY*T(I)*EP/2.0/AX/EXB)*(SXC-POISS*SYC)*
+(ABS(BETAX(I)+BETAX(I)*BETAY(I)-1))
EYC=(-1.0/EP)*(1+LX*T(I)*EP/AY/EYB)*(SYC-POISS*SXC)*
+(ABS(BETAY(I)+BETAY(I)*BETAX(I)-1))
WRITE(10,1280)EXC,EYC
1280 FORMAT('EXC = ',F13.10,' EYC = ',F13.10)

```


C

```

EXT(I)=EXDT+EXC
EYT(I)=EYDT+EYC
EXYT=EXYDT
WRITE(10,1285)EXT(I),EYT(I),EXYT
1285 FORMAT('EXT(I) = ',F13.10,' EYT(I) = ',F13.10,
+' EXYT = ',F13.10)

```

C

```

SXPS(I)=-EP*(EXT(I)+POISS*EYT(I))/(1-POISS**2)
SYPS(I)=-EP*(EYT(I)+POISS*EXT(I))/(1-POISS**2)
SXYP(SI)=-EP*EXYT/(2*(1+POISS))
WRITE(10,1290)SXPS(I),SYPS(I),SXYP(SI)
1290 FORMAT('SXPS(I) = ',F18.3,' SYPS(I) = ',F18.3,
+' SXYP(SI) = ',F18.3)

```

C

```

IF(ITN.GE.2)THEN
SXMOD=SXAV-SXDTI(I)
SYMOD=SYAV-SYDTI(I)
WRITE(10,1363)SXMOD,SYMOD
1363 FORMAT('SXMOD = ',F18.3,' SYMOD = ',F18.3)

```

C

```

SX=-1.0*SXMOD
SY=-1.0*SYMOD
SXY=SXYAV
WRITE(10,1360)SX,SY,SXY
1360 FORMAT('SX = ',F18.3,' SY = ',F18.3,' SXY = ',F18.3)

```

C

```

IF(SX.LT.0.0.AND.SY.LT.0.0)THEN
SXYCR=SXYCR0*((1-SX/SXCR0-SY/SYCR0)**0.5)
ELSE
BB=((SXYCR0**2)*(SX*SYCR0+SY*SXCR0))/(ABS(SXY)*SXCR0*SYCR0)
SXYCR1=(-BB+(BB**2+4*SXYCR0**2)**0.5)/2.0
SXYCR2=(-BB-(BB**2+4*SXYCR0**2)**0.5)/2.0
WRITE(10,1370)BB,SXYCR1,SXYCR2
1370 FORMAT('BB = ',F18.3,/,',SXYCR1 = ',F18.3,/,
+'SXYCR2 = ',F18.3)
IF(ABS(SXYCR1).GE.ABS(SXYCR2))THEN
SXYCR=SXYCR1
ELSE
SXYCR=SXYCR2
ENDIF
ENDIF
SXCR=SX*(ABS(SXYCR)/ABS(SXY))
SYCR=SY*(ABS(SXYCR)/ABS(SXY))
WRITE(10,1380)SXYCR,SXCR,SYCR

```

```

1380  FORMAT('SXYCR= ',F18.3,' SXCR= ',F18.3,' SYCR= ',F18.3)
C
      BUCKT1=SXCR/SXCRO
      BUCKT2=SYCR/SYCRO
      BUCKT3=(SXYCR/SXYCRO)**2
      WRITE(10,1376)BUCKT1,BUCKT2,BUCKT3
1376  FORMAT('BUCKT1 = 'F20.10,/, 'BUCKT2 = 'F20.10,/,
+ 'BUCKT3 = ',F20.10)
      BUCK=SXCR/SXCRO+SYCR/SYCRO+(SXYCR/SXYCRO)**2
      TDB=NINT(BUCK*10000.0)/10000.0
      WRITE(10,1382)BUCK,TDB
1382  FORMAT('Buckling Equation = ',F20.10,/, 'TDB = ',F20.4)
C
      IF(TDB.LT.1.0)THEN
        BPL(I)=BPL(I)+1
      WRITE(10,1384)I
1384  FORMAT(' Panel no',I2,' does not buckle !',/)
      GOTO 3002
    ENDIF
C
      K=TANH(0.5*LOG10(ABS(SXY)/ABS(SXYCR)))
C
      H1=(-K*ABS(SXY))/(EXB*(2.0*AX/LY/T(I)+0.5*(1-K)))
      H2=(-K*ABS(SXY))/(EYB*(AY/LX/T(I)+0.5*(1-K)))
      H3=ABS(SXY)*2.0*K/EP
      H4=(1-K+POISS-POISS*K)*(ABS(SXY)/EP)
      WRITE(10,1400)K,H1,H2,H3,H4
1400  FORMAT('K = ',F7.5,/,
+ 'H1 = ',F15.11,' H2 = ',F15.11,' H3 = ',F15.11,
+ ' H4 = ',F15.11)
C
      XALFA=3.141596/4.0
1450  WRITE(10,1420)XALFA
1420  FORMAT('XALFA = ',F10.7)
      EPSX=H1/TAN(XALFA)
      EPSY=H2*TAN(XALFA)
      EPS=H3/SIN(2*XALFA)+H4*SIN(2*XALFA)
      WRITE(10,1430)EPSX,EPSY,EPS
1430  FORMAT('EPSX = ',F10.8,' EPSY = ',F10.8,' EPS = ',F10.8)
C
      IF((EPS-EPSX).LT.0.0.AND.(EPS-EPSY).GT.0.0)THEN
        PRINT *, ' Diagonal tension angle cannot be calculated !'
        GOTO 3001
      ELSE IF((EPS-EPSX).GT.0.0.AND.(EPS-EPSY).LT.0.0)THEN
        PRINT *, ' Diagonal tension angle cannot be calculated !'

```

```

      GOTO 3001
      ELSE IF((EPS-EPSY).EQ.0.0)THEN
        PRINT *, ' Diagonal tension angle cannot be calculated !'
        GOTO 3001
      ENDIF
      ALFA=ATAN(((EPS-EPSX)/(EPS-EPSY))*0.5)
      ERR=(ABS(ALFA-XALFA))*100.0
      WRITE(10,1440)ALFA,ERR
1440  FORMAT('ALFA = ',F10.7,' % ERR = ',F6.3)
      IF(ERR.GE.0.1)THEN
        XALFA=ALFA
        GOTO 1450
      ENDIF
      WRITE(10,1460)I,ALFA*180.0/3.141596
1460  FORMAT(/,' Diagonal Tension Angle for Panel',I3,' = ',F6.3,
+ ' degrees !',/)
C
      SXDT(I)=K*ABS(SXY)/TAN(ALFA)
      SYDT(I)=K*ABS(SXY)*TAN(ALFA)
      WRITE(10,1500)SXDT(I),SYDT(I)
1500  FORMAT('SXDT(I) = ',F18.3,' SYDT(I) = ',F18.3)
C
C           Tests 1 to 5 in the Grisham algorithm
C
      BBETAX=0.0
      BBETAY=0.0
C
2400  EXDT=(-1.0/EP)*(EP*LY*T(I)/2.0/AX/EXB+1)*(SXDT(I)-(1-BBETAX)*
+ SXDTI(I)-POISS*(SYDT(I)-(1-BBETAX)*SYDTI(I)))
      EYDT=(-1.0/EP)*(EP*LX*T(I)/AY/EYB+1)*(SYDT(I)-(1-BBETAY)*
+ SYDTI(I)-POISS*(SXDT(I)-(1-BBETAY)*SXDTI(I)))
      WRITE(10,2540)EXDT,EYDT
2540  FORMAT('EXDT = ',F13.10,' EYDT = ',F13.10)
C
      THETA=(1.0/(2*(1.0+POISS)))*(4.0/(SIN(2*ALFA))**2+
+ (EP*LX*T(I)*(TAN(ALFA))**2)/(EYB*(AY+(0.5*LX*T(I))*(1-K)))+
+ (EP*LY*T(I)/(TAN(ALFA))**2)/(EXB*(2.0*AX+(0.5*LY*T(I))*(1-K))))
      EXYDT=K*(THETA-1)*SXY/GP
      WRITE(10,2560)THETA,EXYDT
2560  FORMAT('THETA = ',F10.6,' EXYDT = ',F13.10)
C
      C1=1+LY*T(I)*EP*BETAX(I)/2.0/AX/EXB
      C2=1+LX*T(I)*EP*BETAY(I)/AY/EYB
      C3=1-BETAX(I)
      C4=1-BETAY(I)

```

```

        WRITE(10,2580)C1,C2,C3,C4,BETAX(I),BETAY(I)
2580  FORMAT('C1 = ',F10.6,' C2 = ',F10.6,' C3 = ',F10.6,
        +' C4 = ',F10.6,/, 'BETAX(I) = ',F10.6,' BETAY(I) = ',F10.6)
C
        SXC=(C2*C3*(SX-SXCR)*L+BETAX(I)*POISS*C4*(SY-SYCR)*L)
        +/C1/C2
        SYC=(C1*C4*(SY-SYCR)*L+BETAY(I)*POISS*C3*(SX-SXCR)*L)
        +/C1/C2
        WRITE(10,2600)SXC,SYC
2600  FORMAT('SXC = ',F18.3,' SYC = ',F18.3)
C
        EXC=(-1.0/EP)*(1+LY*T(I)*EP/2.0/AX/EXB)*(SXC-POISS*SYC)*
        +(ABS(BETAX(I)+BETAX(I)*BETAY(I)-1))
        EYC=(-1.0/EP)*(1+LX*T(I)*EP/AY/EYB)*(SYC-POISS*SXC)*
        +(ABS(BETAY(I)+BETAX(I)*BETAY(I)-1))
        WRITE(10,2620)EXC,EYC
2620  FORMAT('EXC = ',F13.10,' EYC = ',F13.10)
C
        EXT(I)=(1-BBETAX)*EXT(I)+EXDT+EXC
        EYT(I)=(1-BBETAY)*EYT(I)+EYDT+EYC
        EXYT=EXYDT
        WRITE(10,2640)EXT(I),EYT(I),EXYT
2640  FORMAT('EXT = ',F13.10,' EYT = ',F13.10,' EXYT = 'F13.10)
C
        SXPS(I)=-EP*(EXT(I)+POISS*EYT(I))/(1-POISS**2)
        SYPS(I)=-EP*(EYT(I)+POISS*EXT(I))/(1-POISS**2)
        SXYP(SI)=-EP*EXYT/(2*(1+POISS))
C
        WRITE(10,2660)SXPS(I),SYPS(I),SXYP(SI)
2660  FORMAT('SXPS(I) = ',F18.3,' SYPS(I) = ',F18.3,
        +' SXYP(SI) = ',F18.3)
        ENDIF
C
        calculate convergence parameters
C
        WRITE(10,2680)SXC,SXCR,SYC,SYCR
2680  FORMAT('SXC = ',F18.3,' SXCR = ',F18.3,/,
        +'SYC = ',F18.3,' SYCR = ',F18.3)
        IF(ABS(SXCR).LT.1.0E-5)THEN
        WRITE(21,2682)
2682  FORMAT(' K1X --> INF ; K1XP --> 0')
        ELSE IF(ABS(SXC).LT.1.0E-5)THEN
        WRITE(21,2684)
2684  FORMAT(' K1X --> 0 ; K1XP --> INF')
        ELSE

```

```

      K1X=SXC/SXCR
      K1XP=(K1X-K1XN(I))*100.0/K1X
      K1XPP=(K1XN(I)-K1XNN(I))*100.0/K1XN(I)
      WRITE(10,2687)K1X,K1XP,K1XPP
2687  FORMAT('K1X = ',F10.3,5X,'K1XP = ',F10.3,5X,'K1XPP = ',
+         F10.3)
      ENDIF
C
      IF(ABS(SYCR).LT.1.0E-5)THEN
      WRITE(21,2686)
2686  FORMAT(' K1Y --> INF ; K1YP --> 0')
      ELSE IF(ABS(SYC).LT.1.0E-5)THEN
      WRITE(21,2688)
2688  FORMAT(' K1Y --> 0 ; K1YP --> INF')
      ELSE
      K1Y=SYC/SYCR
      K1YP=(K1Y-K1YN(I))*100.0/K1Y
      K1YPP=(K1YN(I)-K1YNN(I))*100.0/K1YN(I)
      WRITE(10,2689)K1Y,K1YP,K1YPP
2689  FORMAT('K1Y = ',F10.3,5X,'K1YP = ',F10.3,5X,'K1YPP = ',
+         F10.3)
      ENDIF
C
      FN=I+30
      WRITE(FN,2662)IBC,ITN,K1X,SXC,SXCR,SXYCR,SXY,SX-SXCR,SY-SYCR,
+         SX,SXAV,SXDT(I)
2662  FORMAT(2I3,10G12.5)
C
      FN2=I+40
      WRITE(FN2,2663)IBC,ITN,K1Y,SYC,SYCR,SXYCR,SXY,SY-SYCR,SX-SXCR,
+         SY,SYAV,SYDT(I)
2663  FORMAT(2I3,10G12.5)
C
2704  WRITE(10,2700)SXDT(I),SXDTI(I),SYDT(I),SYDTI(I)
2700  FORMAT('SXDT(I) = ',F18.3,' SXDTI(I) = ',F18.3,/,
+         'SYDT(I) = ',F18.3,' SYDTI(I) = ',F18.3)
      K2X=SXDT(I)-SXDTI(I)
      K2Y=SYDT(I)-SYDTI(I)
      K2XP=(K2X/SXDT(I))*100.0
      K2YP=(K2Y/SYDT(I))*100.0
      K2XPP=(K2XN(I)/SXDTI(I))*100.0
      K2YPP=(K2YN(I)/SYDTI(I))*100.0
      K2XPPP=(K2XNN(I)/SXDTII(I))*100.0
      K2YPPP=(K2YNN(I)/SYDTII(I))*100.0
      WRITE(10,2710)I,K1X,K1XP,K1Y,K1YP,K1XPP,K1YPP,

```

```

+K2X,K2XP,K2Y,K2YP,K2XPP,K2YPP,K2XPPP,K2YPPP
  WRITE(21,2710)I,K1X,K1XP,K1Y,K1YP,K1XPP,K1YPP,
+K2X,K2XP,K2Y,K2YP,K2XPP,K2YPP,K2XPPP,K2YPPP
2710  FORMAT('Panel No ',I2,/, 'K1X = ',F18.3, '( ',F10.3, ' %)',
+5X, 'K1Y = ',F18.3, '( ',F10.3, ' %)',/,24X, '( ',F10.3, ' %)',
+29X, '( ',F10.3, ' %)',/,
+'K2X = ',F18.3, '( ',F10.3, ' %)',5X, 'K2Y = ',
+F18.3, '( ',F10.3, ' %)',/,24X, '( ',F10.3, ' %)',29X,
+' ( ',F10.3, ' %)',/,24X, '( ',F10.3, ' %)',29X,
+' ( ',F10.3, ' %)'')

```

C

```

  SXCRI(I)=SXCR
  SYCRI(I)=SYCR
  SXDTII(I)=SXDTI(I)
  SYDTII(I)=SYDTI(I)
  SXDTI(I)=SXDT(I)
  SYDTI(I)=SYDT(I)
  WRITE(10,2720)SXCRI(I),SYCRI(I),SXDTI(I),SYDTI(I),
+SXDTII(I),SYDTII(I)
2720  FORMAT('SXCRI(I) = ',F18.3, '      SYCRI(I) = ',F18.3,/,
+'SXDTI(I) = ',F18.3, '      SYDTI(I) = ',F18.3,/,
+'SXDTII(I) = ',F18.3, '      SYDTII(I) = ',F18.3)
  K1XNN(I)=K1XN(I)
  K1YNN(I)=K1YN(I)
  K1XN(I)=K1X
  K1YN(I)=K1Y
  K2XNN(I)=K2XN(I)
  K2YNN(I)=K2YN(I)
  K2XN(I)=K2X
  K2YN(I)=K2Y

```

C

```

  WRITE(10,2726)K1XNN(I),K1YNN(I),K1XN(I),K1YN(I)
2726  FORMAT('K1XNN(I) = ',F10.3,5X, 'K1YNN(I) = ',F10.3,/,
+'K1XN(I) = ',F10.3,5X, 'K1YN(I) = ',F10.3)
  WRITE(10,2728)K2XNN(I),K2YNN(I),K2XN(I),K2YN(I)
2728  FORMAT('K2XNN(I) = ',F18.3,5X, 'K2YNN(I) = ',F18.3,/,
+'K2XN(I) = ',F18.3,5X, 'K2YN(I) = ',F18.3)

```

C

C Test for convergence:

C

```

  IF(ITN.GE.3)THEN
  IF(ABS(K1XP).LT.5.0.AND.ABS(K1XPP).LT.5.0)THEN
  WRITE(13,2730)K1X,I,ITN
2730  FORMAT(4X, 'K1X converges to',F10.3, ' for plate ',I3,
+ ' !! (ITN=',I3,')')

```

```

      K1XVAL(I)=K1X
      CONA(I)=1
      ELSE IF(CONA(I).NE.1)THEN
        K1XNC(I)=K1X
        K1XID(I)=1
        WRITE(13,2735)K1XNC(I),K1XID(I),I
2735  FORMAT('K1XNC(I) =',F15.7,'      K1XID(I) =',I3,'      I =',I3)
      ENDIF
      IF(ABS(K1YP).LT.5.0.AND.ABS(K1YPP).LT.5.0)THEN
        WRITE(13,2740)K1Y,I,ITN
2740  FORMAT(4X,'K1Y converges to',F10.3,' for plate ',I3,
+ ' !! (ITN=',I3,')')
        K1YVAL(I)=K1Y
        CONB(I)=1
        ELSE IF(CONB(I).NE.1)THEN
          K1YNC(I)=K1Y
          K1YID(I)=1
          WRITE(13,2745)K1YNC(I),K1YID(I),I
2745  FORMAT('K1YNC(I) =',F15.7,'      K1YID(I) =',I3,'      I =',I3)
        ENDIF
        IF(ABS(K2XP).LT.2.0.AND.ABS(K2XPP).LT.2.0.AND.ABS(K2XPPP).
+LT.2.0)THEN
          WRITE(13,2780)I,ITN
2780  FORMAT(4X,'K2X converges for plate ',I3,' !! (ITN=',
+ I3,')')
          CONC(I)=1
          ENDIF
          IF(ABS(K2YP).LT.2.0.AND.ABS(K2YPP).LT.2.0.AND.ABS(K2YPPP).
+LT.2.0)THEN
            WRITE(13,2790)I,ITN
2790  FORMAT(4X,'K2Y converges for plate ',I3,' !! (ITN=',
+ I3,')')
            COND(I)=1
            ENDIF
            ENDIF
            CONAT=CONAT+CONA(I)
            CONBT=CONBT+CONB(I)
            CONCT=CONCT+CONC(I)
            CONDT=CONDT+COND(I)
3002  CONTINUE
C
      IF(CONAT+CONBT+CONCT+CONDT.EQ.4*NP)THEN
        NOIT=ITN
        WRITE(13,3003)NOIT
3003  FORMAT('NOIT = ',I3)

```

```

        GOTO 3006
    ELSE
        NOIT=ITN
    ENDIF
C
3000 CONTINUE
C
3006 WRITE(13,3007)ITN
3007 FORMAT('ITN = ',I5)
        BCTOT=0
        DO 3005 I=1,NP
            BCONA(I)=0
            BCONB(I)=0
3005 CONTINUE
C
        IF(IBC.EQ.IBCMAX)THEN
            WRITE(13,3206)IBC
3206  FORMAT('Maximum number of iterations reached - ',I3,' - still',
+ ' no convergence !')
            GOTO 3001
        ENDIF
C
        DO 3051 I=1,NP
C
C            x-BETA values
C
C            modify the BETAs when the solution does not buckle
C
            IF(BPL(I).EQ.NOIT)THEN
                WRITE(13,3011)I,BETAX(I)
3011  FORMAT('Panel no',I3,' converges to an unbuckled state !',/,
+ 'The BETAX value will therefore remain as is:',/,
+ 'BETAX(I) = ',F10.7)
                ELSE IF(BPL(I).LT.NOIT.AND.BPL(I).GT.0.AND.CONA(I).EQ.0)THEN
                    WRITE(13,3012)I,BPL(I)
3012  FORMAT('Panel number',I3,' buckled less than 12 times (BPL=',
+ I3,');',/, 'probably does not converge to an unbuckled state !')
                    WRITE(13,3013)I,BETAX(I)
3013  FORMAT('Old BETAX(I) value for panel no',I3,' is:',F10.7)
                    BETAX(I)=BETAX(I)+0.02
                    IF(BETAX(I).GE.1.0)BETAX(I)=0.001
                    WRITE(13,3014)I,BETAX(I)
3014  FORMAT('New BETAX(I) value for panel no',I3,' is:',F10.7)
C
C            modify the BETAs when the solution does not converge

```


C

```

ELSE IF(K1XID(I).EQ.1.AND.K1XNC(I).LT.1.0)THEN
WRITE(13,3050)I,BETAX(I)
3050  FORMAT('K1X of panel no',I3,' did not converge !',/,
+ 'Old BETAX(I) value for panel',I3,' is: ',F10.7)
BETAX(I)=BETAX(I)-0.016
IF(BETAX(I).LE.0.0)BETAX(I)=0.001
WRITE(13,3055)I,BETAX(I)
3055  FORMAT('New BETAX(I) value for panel',I3,' is: ',F10.7)
ELSE IF(K1XID(I).EQ.1.AND.K1XNC(I).GT.1.0)THEN
WRITE(13,3060)I,I,BETAX(I)
3060  FORMAT('K1X of panel no',I3,' did not converge !',/,
+ 'Old BETAX(I) value for panel',I3,' is: ',F10.7)
BETAX(I)=BETAX(I)+0.02
IF(BETAX(I).GE.1.0)BETAX(I)=0.999
WRITE(13,3065)I,BETAX(I)
3065  FORMAT('New BETAX(I) value for panel',I3,' is: ',F10.7)

```

C

```

C      modify the BETAs when the solution converges

```

C

```

ELSE IF(CONA(I).EQ.1.AND.K1XVAL(I).LT.0.8)THEN
WRITE(13,3015)I,BETAX(I)
3015  FORMAT('Old BETAX(I) value for panel',I3,' is: ',F10.7)
BETAX(I)=BETAX(I)-0.02
WRITE(13,3020)I,BETAX(I)
3020  FORMAT('New BETAX(I) value for panel',I3,' is: ',F10.7)
ELSE IF(CONA(I).EQ.1.AND.K1XVAL(I).GE.0.8.AND.
+ K1XVAL(I).LT.0.95)THEN
WRITE(13,3021)I,BETAX(I)
3021  FORMAT('Old BETAX(I) value for panel',I3,' is: ',F10.7)
BETAX(I)=BETAX(I)-0.002
WRITE(13,3022)I,BETAX(I)
3022  FORMAT('New BETAX(I) value for panel',I3,' is: ',F10.7)
ELSE IF(CONA(I).EQ.1.AND.K1XVAL(I).GT.1.2)THEN
WRITE(13,3025)I,BETAX(I)
3025  FORMAT('Old BETAX(I) value for panel',I3,' is: ',F10.7)
BETAX(I)=BETAX(I)+0.02
WRITE(13,3030)I,BETAX(I)
3030  FORMAT('New BETAX(I) value for panel',I3,' is: ',F10.7)
ELSE IF(CONA(I).EQ.1.AND.K1XVAL(I).GT.1.05.AND.
+ K1XVAL(I).LE.1.2)THEN
WRITE(13,3031)I,BETAX(I)
3031  FORMAT('Old BETAX(I) value for panel',I3,' is: ',F10.7)
BETAX(I)=BETAX(I)+0.002
WRITE(13,3032)I,BETAX(I)

```

```

3032   FORMAT('New BETAX(I) value for panel',I3,' is: ',F10.7)
      ELSE IF(CONA(I).EQ.1.AND.K1XVAL(I).GE.0.95.AND.
+ K1XVAL(I).LE.1.05)THEN
      BCONA(I)=1
      WRITE(13,3040)I,BETAX(I),BCONA(I)
3040   FORMAT('BETAX(I) is the correct value for K1X to converge',
+ ' for plate ',I3,'(BETAX(I) = ',F10.7,')',/, 'BCONA = ',I3)
C
C       modify BETAs when none of the above apply
C
      ELSE
      WRITE(13,3041)I,BETAX(I)
3041   FORMAT('Default change - Old BETAX(I) value for panel',I3,
+ ' is: ',F10.7)
      BETAX(I)=BETAX(I)+0.03
      IF(BETAX(I).GE.1.0)BETAX(I)=0.001
      WRITE(13,3044)I,BETAX(I)
3044   FORMAT('Default change - New BETAX(I) value for panel',I3,
+ ' is: ',F10.7)
      ENDIF
C
C       y-BETA values
C
C       modify the BETAs when the solution does not buckle
C
      IF(BPL(I).EQ.NOIT)THEN
      WRITE(13,2791)I,BETAY(I)
2791   FORMAT('Panel no',I3,' converges to an unbuckled state !',/,
+ 'The BETAY value will therefore remain as is:',/,
+ 'BETAY(I) = ',F10.7)
      BTOT=BTOT+1
      BPID(I)=1
      ELSE IF(BPL(I).LT.NOIT.AND.BPL(I).GT.0.AND.CONB(I).EQ.0)THEN
      WRITE(13,3128)I,BPL(I)
3128   FORMAT('Panel number',I3,' buckled less than 12 times (BPL=',
+ I3,');',/, 'probably does not converge to an unbuckled state !')
      WRITE(13,3129)I,BETAY(I)
3129   FORMAT('Old BETAY(I) value for panel no',I3,' is:',F10.7)
      BETAY(I)=BETAY(I)+0.02
      IF(BETAY(I).GE.1.0)BETAY(I)=0.001
      WRITE(13,3131)I,BETAY(I)
3131   FORMAT('New BETAY(I) value for panel no',I3,' is:',F10.7)
C
C       modify BETAs when solution does not converge
C

```

```

ELSE IF(K1YID(I).EQ.1.AND.K1YNC(I).LT.1.0)THEN
  WRITE(13,3110)I,I,BETAY(I)
3110  FORMAT('K1Y of panel no',I3,' did not converge !',/,
+ 'Old BETAY(I) value for panel',I3,' is: ',F10.7)
  BETAY(I)=BETAY(I)-0.016
  IF(BETAY(I).LE.0.0)BETAY(I)=0.001
  WRITE(13,3115)I,BETAY(I)
3115  FORMAT('New BETAY(I) value for panel',I3,' is: ',F10.7)
ELSE IF(K1YID(I).EQ.1.AND.K1YNC(I).GT.1.0)THEN
  WRITE(13,3120)I,I,BETAY(I)
3120  FORMAT('K1Y of panel no',I3,' did not converge !',/,
+ 'Old BETAY(I) value for panel',I3,' is: ',F10.7)
  BETAY(I)=BETAY(I)+0.02
  IF(BETAY(I).GE.1.0)BETAY(I)=0.999
  WRITE(13,3125)I,BETAY(I)
3125  FORMAT('New BETAY(I) value for panel',I3,' is: ',F10.7)
C
C      modify BETAs when solution converges
C
ELSE IF(CONB(I).EQ.1.AND.K1YVAL(I).LT.0.8)THEN
  WRITE(13,3075)I,BETAY(I)
3075  FORMAT('Old BETAY(I) value for panel',I3,' is: ',F10.7)
  BETAY(I)=BETAY(I)-0.02
  WRITE(13,3080)I,BETAY(I)
3080  FORMAT('New BETAY(I) value for panel',I3,' is: ',F10.7)
ELSE IF(CONB(I).EQ.1.AND.K1YVAL(I).GE.0.8.AND.
+ K1YVAL(I).LT.0.95)THEN
  WRITE(13,3081)I,BETAY(I)
3081  FORMAT('Old BETAY(I) value for panel',I3,' is: ',F10.7)
  BETAY(I)=BETAY(I)-0.002
  WRITE(13,3082)I,BETAY(I)
3082  FORMAT('New BETAY(I) value for panel',I3,' is: ',F10.7)
ELSE IF(CONB(I).EQ.1.AND.K1YVAL(I).GT.1.2)THEN
  WRITE(13,3085)I,BETAY(I)
3085  FORMAT('Old BETAY(I) value for panel',I3,' is: ',F10.7)
  BETAY(I)=BETAY(I)+0.02
  WRITE(13,3090)I,BETAY(I)
3090  FORMAT('New BETAY(I) value for panel',I3,' is: ',F10.7)
ELSE IF(CONB(I).EQ.1.AND.K1YVAL(I).GT.1.05.AND.
+ K1YVAL(I).LE.1.2)THEN
  WRITE(13,3091)I,BETAY(I)
3091  FORMAT('Old BETAY(I) value for panel',I3,' is: ',F10.7)
  BETAY(I)=BETAY(I)+0.002
  WRITE(13,3092)I,BETAY(I)
3092  FORMAT('New BETAY(I) value for panel',I3,' is: ',F10.7)

```

```

        ELSE IF(CONB(I).EQ.1.AND.K1YVAL(I).GE.0.95.AND.
+ K1YVAL(I).LE.1.05)THEN
        BCONB(I)=1
        WRITE(13,3100)I,BETAY(I),BCONB(I)
3100  FORMAT('BETAY(I) is the correct value for K1Y to converge',
+ ' for panel ',I3,'(BETAY(I) = ',F10.7,')',/, 'BCONB = ',I3)
C
C      modify BETAs when none of the above apply
C
        ELSE
        WRITE(13,3101)I,BETAY(I)
3101  FORMAT('Default change - Old BETAY(I) value for panel',I3,
+ ' is: ',F10.7)
        BETAY(I)=BETAY(I)+0.03
        IF(BETAY(I).GE.1.0)BETAY(I)=0.001
        WRITE(13,3104)I,BETAY(I)
3104  FORMAT('Default change - New BETAY(I) value for panel',I3,
+ ' is: ',F10.7)
        ENDIF
        BCTOT=BCTOT+BCONA(I)+BCONB(I)
        WRITE(13,3180)BTOT,BCTOT
3180  FORMAT('BTOT = ',I4,'      BCTOT = ',I4)
3051  CONTINUE
C
        IF(BTOT.NE.NP.AND.BCTOT.EQ.2*(NP-BTOT))THEN
        WRITE(13,3512)
3512  FORMAT('BETAs adjusted successfully - all no 1 convergence',
+ ' criteria is satisfied !')
        IF(CONCT.EQ.NP)THEN
        WRITE(13,3513)
3513  FORMAT('The Diagonal tension stress requirement in the x-dir',/,
+ 'is satisfied (no 2 requirement)')
        ELSE
        WRITE(13,3514)
3514  FORMAT('Requirement no 2 is not satisfied in x-dir !')
        ENDIF
        IF(CONDT.EQ.NP)THEN
        WRITE(13,3516)
3516  FORMAT('The diagonal tension stress requirement in the y-dir',/,
+ 'is satisfied (no 2 requirement)')
        ELSE
        WRITE(13,3517)
3517  FORMAT('Requirement no 2 is not satisfied in y-dir !')
        ENDIF
        GOTO 3001

```

```

        ELSE IF(BTOT.EQ.NP)THEN
          WRITE(13,3518)
3518   FORMAT('No panels buckled !')
          GOTO 3001
        ENDIF
C
3500  CONTINUE
C
        CALL TIME(SEC)
        TSTOP=SEC
        WRITE(10,3505)TSTOP
3505  FORMAT('TSTOP = ',I20)
C
        CLOSE(10)
        CLOSE(20)
        CLOSE(21)
        CLOSE(13)
        CLOSE(12)
        CLOSE(31)
        CLOSE(32)
        CLOSE(33)
        CLOSE(34)
        CLOSE(35)
        CLOSE(36)
        CLOSE(41)
        CLOSE(42)
        CLOSE(43)
        CLOSE(44)
        CLOSE(45)
        CLOSE(46)
C
        STOP
        END

C*****
C
        SUBROUTINE FNODES(NPX,NPY,NEX,NEY,C20,LTOTX,LTOTY,UPRE,
+SENODE,SNALL,SNCX,SNCY,SNCZ,SNTOT,SNTFLAT)
C
C*****
C          Subroutine to generate all nodes for the

```

```

C          finite element model
C*****
C
      DOUBLE PRECISION LTOTX,LTOTY,LPNX,LPNY,
+SNCX(1000),SNCY(1000),SNCZ(1000),SENODE(301)
      INTEGER YINC,SNALL(1000),C20,NPX,NPY,NEX,NEY,SNTOT,SNTFLAT
      CHARACTER*(3)UPRE

C
      OPEN(20,FILE='nodedata.dat')

C
      NNXMAX=C20*NPX*NEX+1
      NONX=NNXMAX
      NONY=C20*NPY*NEY+1
      NIPPX=C20*NEX
      NBEPP=2*NEX+2*NEY
      NSEPP=NEX*NEY
      LPNX=LTOTX/(NONX-1)
      LPNY=LTOTY/(NONY-1)
      NINCY=NNXMAX
      NNYMAX=C20*NPY*NEY+1
      NTOTAL=NNXMAX*NNYMAX
      NIPPY=C20*NEY*NINCY
      WRITE(10,30)NNXMAX,NNYMAX,NONX,NONY,NIPPX,NIPPY,
+ NBEPP,NSEPP,LPNX,LPNY
30   FORMAT('NNXMAX = ',I5,/, 'NNYMAX = ',I5,/,
+ 'NONX = ',I5,/, 'NONY = ',I5,/, 'NIPPX = ',I5,/, 'NIPPY = ',I5,
+ /, 'NBEPP = ',I5,/, 'NSEPP = ',I5,/, 'LPNX = ',F10.4,/,
+ 'LPNY = ',F10.4)

C
C          generate nodes in one plane
C
      NCOUNT=0
      YINC=1
      DO 120 J=1,NONY
      DO 100 I=1,NONX
      K=1
      IF(J.EQ.1)THEN
      NCOUNT=NCOUNT+1
      SNALL(NCOUNT)=NCOUNT
      SNCX(NCOUNT)=0.0+LPNX*(I-1)
      SNCY(NCOUNT)=0.0
      SNCZ(NCOUNT)=0.0
      WRITE(20,101)SNALL(NCOUNT),I,J,NCOUNT,YINC,SNCX(NCOUNT),
+ SNCY(NCOUNT),SNCZ(NCOUNT)
101  FORMAT('SNALL(NCOUNT) = ',I4,' I = ',I4,' J = ',I4,

```

```

+   ' NCOUNT = ',I4,' YINC = ',I4,' SNCX(NCOUNT) = ',F10.4,
+   ' SNCY(NCOUNT) = ',F10.4,' SNCZ(NCOUNT) = ',F10.4)
  GOTO 100
  ELSE
    NCOUNT=NCOUNT+1
    SMALL(NCOUNT)=NCOUNT
    SNCX(NCOUNT)=0.0+LPNX*(I-1)
    SNCY(NCOUNT)=SNCY(NCOUNT)+(J-1)*LPNY
    SNCZ(NCOUNT)=0.0
    WRITE(20,102)SMALL(NCOUNT),I,J,NCOUNT,YINC,SNCX(NCOUNT),
+   SNCY(NCOUNT),SNCZ(NCOUNT)
102  FORMAT('SMALL(NCOUNT) = ',I4,' I = ',I4,' J = ',I4,
+   ' NCOUNT = ',I4,' YINC = ',I4,' SNCX(NCOUNT) = ',F10.4,
+   ' SNCY(NCOUNT) = ',F10.4,' SNCZ(NCOUNT) = ',F10.4)
  ENDIF
100  CONTINUE
    YINC=NINCY
120  CONTINUE
    SNTFLAT=NCOUNT
C
C     generate nodes in second plane for upright eccentricity
C
  IF(UPRE.EQ.'ECC')THEN
    YINC=1
    DO 115 J=1,NONY
      NUPRC=0
      DO 110 I=1,NONX,C20*NEX
        NUPRC=NUPRC+1
        IF(J.EQ.1)THEN
          NCOUNT=NCOUNT+1
          SMALL(NCOUNT)=NCOUNT
          SNCX(NCOUNT)=0.0+LPNX*(I-1)
          SNCY(NCOUNT)=0.0
          SNCZ(NCOUNT)=SENODE(NUPRC)
          WRITE(20,105)SMALL(NCOUNT),I,J,NCOUNT,YINC,SNCX(NCOUNT),
+   SNCY(NCOUNT),SNCZ(NCOUNT)
105  FORMAT('SMALL(NCOUNT) = ',I4,' I = ',I4,' J = ',I4,
+   ' NCOUNT = ',I4,' YINC = ',I4,' SNCX(NCOUNT) = ',F10.4,
+   ' SNCY(NCOUNT) = ',F10.4,' SNCZ(NCOUNT) = ',F10.4)
          GOTO 110
        ELSE
          NCOUNT=NCOUNT+1
          SMALL(NCOUNT)=NCOUNT
          SNCX(NCOUNT)=0.0+LPNX*(I-1)
          SNCY(NCOUNT)=SNCY(NCOUNT)+(J-1)*LPNY

```

```

        SNCZ(NCOUNT)=SENODE(NUPRC)
        WRITE(20,106)SNALL(NCOUNT),I,J,NCOUNT,YINC,SNCX(NCOUNT),
+       SNCY(NCOUNT),SNCZ(NCOUNT)
106     FORMAT('SNALL(NCOUNT) = ',I4,' I = ',I4,' J = ',I4,
+       ' NCOUNT = ',I4,' YINC = ',I4,' SNCX(NCOUNT) = ',F10.4,
+       ' SNCY(NCOUNT) = ',F10.4,' SNCZ(NCOUNT) = ',F10.4)
        ENDIF
110     CONTINUE
        YINC=NINCY
115     CONTINUE
        SNTOT=NCOUNT
        PRINT *, 'SNTOT = ',SNTOT
    ELSE
        SNTOT=NCOUNT
        PRINT *, 'SNTOT = ',SNTOT
    ENDIF
C
        CLOSE(20)
        RETURN
    END

```

```

C*****
C
        SUBROUTINE ELEMENTS(NPX,NPY,NEX,NEY,C20,UPRE,NALL,
+SELS,SELBH,SELBV,SELBVN,SELBHN,SELSN,SPNLSE,
+ICSS,ICBHS,ICBVS,NTFLAT)
C
C*****
C        Subroutine to generate all elements for the
C        finite element model
C*****
C
        INTEGER NALL(1000),NPX,NPY,NEX,NEY,C20,NNXMAX,
+SELS(1000,1000),SELBH(1000,1000),SELBV(1000,1000),
+SELBVN(1000),SELBHN(1000),SELSN(1000),SPNLSE(1000,20),
+ICSS,ICBHS,ICBVS,NTFLAT
        CHARACTER*(3) UPRE
        PRINT *, 'NPX = ',NPX
        PRINT *, 'NPY = ',NPY
        PRINT *, 'NEX = ',NEX
        PRINT *, 'NEY = ',NEY

```



```

PRINT *, 'C20 =', C20
PRINT *, 'UPRE =', UPRE
C
OPEN(20, FILE='element-data.dat')
C
NNXMAX=C20*NPX*NEX+1
NONX=NNXMAX
NONY=C20*NPY*NEY+1
C
C      generate the shell elements
C
ICSS=0
NINC=0
DO 169 I=1, NEY*NPY, 1
NINC=(I-1)*NONX*C20
DO 171 NCOUNT=1+NINC, (NONX-C20)+NINC, C20
ICSS=ICSS+1
SELSN(ICSS)=ICSS
SELS(ICSS, 1)=NALL(NCOUNT)
SELS(ICSS, 2)=NALL(NCOUNT+C20)
SELS(ICSS, 3)=NALL(NCOUNT+NONX*C20+C20)
SELS(ICSS, 4)=NALL(NCOUNT+NONX*C20)
SELS(ICSS, 5)=NALL(NCOUNT+C20/2)
SELS(ICSS, 6)=NALL(NCOUNT+NONX+C20)
SELS(ICSS, 7)=NALL(NCOUNT+NONX*C20+C20/2)
SELS(ICSS, 8)=NALL(NCOUNT+NONX)
WRITE(20, 181) ICSS, SELS(ICSS, 1), ICSS, SELS(ICSS, 2), ICSS,
+ SELS(ICSS, 3),
+ ICSS, SELS(ICSS, 4), ICSS, SELS(ICSS, 5), ICSS, SELS(ICSS, 6), ICSS,
+ SELS(ICSS, 7), ICSS, SELS(ICSS, 8), I
181  FORMAT('SELS(', I3, ', ', 1) = ', I5, ' SELS(', I3, ', ', 2) = ', I5,
+ ' SELS(', I3, ', ', 3) = ', I5, ' SELS(', I3, ', ', 4) = ', I5, /,
+ 'SELS(', I3, ', ', 5) = ', I5, ' SELS(', I3, ', ', 6) = ', I5,
+ ' SELS(', I3, ', ', 7) = ', I5, ' SELS(', I3, ', ', 8) = ', I5, /,
+ 'I = ', I3)
171  CONTINUE
169  CONTINUE
C
C      create element sets (shells) related to panels/sheets
C
IC4=0
IC5=0
WRITE(20, *) ' '
DO 530 IL=1, NEY
IC3=0

```

```

DO 505 I=1,NEY*NPY,NEY
DO 500 J=1,NEX*NPX,NEX
  IC3=IC3+1
  DO 510 IK=1,NEX
    IC4=IK+NEX*(IL-1)
    IC5=IC5+1
    SPNLSE(IC3,IC4)=SELSN(IC5)
    WRITE(20,520)IC3,IC4,SPNLSE(IC3,IC4)
520    FORMAT(1X,'SPNLSE(',I2,',',I2,',') = ',I3)
510    CONTINUE
500    CONTINUE
    IC5=IC5+NEX*NPX*(NEY-1)
505    CONTINUE
    IC5=NEX*NPX*IL
530    CONTINUE
C
C      generate the beam elements - horizontal
C
  NINC=0
  ICBHS=0
  ICB=ICSS
  DO 194 I=1,NPY+1,1
    NINC=(I-1)*NONX*C20*NEY
    DO 192 NCOUNT=1+NINC,NONX-C20+NINC,C20
      ICB=ICB+1
      ICBHS=ICBHS+1
      SELBHN(ICBHS)=ICB
      SELBH(ICBHS,1)=NALL(NCOUNT)
      SELBH(ICBHS,2)=NALL(NCOUNT+C20/2)
      SELBH(ICBHS,3)=NALL(NCOUNT+C20)
      WRITE(20,191)ICBHS,SELBHN(ICBHS),ICBHS,SELBH(ICBHS,1),ICBHS,
+ SELBH(ICBHS,2),ICBHS,SELBH(ICBHS,3),I,J
191    FORMAT('SELBHN(',I3,',') = ',I5,' SELBH(',I3,',1) = ',I5,
+ ' SELBH(',I3,',2) = ',I5,' SELBH(',I3,',3) = ',I5,/,
+ 'I = ',I5,' J = ',I5)
192    CONTINUE
194    CONTINUE
C
C      generate the beam elements - vertical - no eccentricity
C
  IF(UPRE.EQ.'NOE')THEN
  ICBVS=0
  NINC=0
  DO 210 I=1,NPX+1,1
    NINC=(I-1)*NEX*C20

```

```

DO 220 NCOUNT=1+NINC, NONX*NEY*C20+NINC, NONX*C20
  ICB=ICB+1
  ICBVS=ICBVS+1
  SELBVN(ICBVS)=ICB
  SELBV(ICBVS,1)=NALL(NCOUNT)
  SELBV(ICBVS,2)=NALL(NCOUNT+NONX)
  SELBV(ICBVS,3)=NALL(NCOUNT+NONX*C20)
  WRITE(20,211) ICBVS, SELBVN(ICBVS), ICBVS, SELBV(ICBVS,1), ICBVS,
+   SELBV(ICBVS,2), ICBVS, SELBV(ICBVS,3), I, J, ICB
211  FORMAT('SELBVN(',I3,') = ',I5,' SELBV(',I3,',1) = ',I5,
+   ' SELBV(',I3,',2) = ',I5,' SELBV(',I3,',3) = ',I5,/,
+   'I = ',I5,' J = ',I5,' ICB = ',I5)
220  CONTINUE
210  CONTINUE
C
C      generate the beam elements - vertical - with eccentricity
C
  ELSE IF(UPRE.EQ.'ECC') THEN
    ICBVS=0
    NINC=0
    DO 240 I=1,NPX+1,1
      NINC=I-1
      DO 235 NCOUNT=NTFLAT+1+NINC, NTFLAT+NPY*NEY*C20*(NPX+1)+NINC,
+ (NPX+1)*C20
        ICB=ICB+1
        ICBVS=ICBVS+1
        SELBVN(ICBVS)=ICB
        SELBV(ICBVS,1)=NALL(NCOUNT)
        SELBV(ICBVS,2)=NALL(NCOUNT+NPX+1)
        SELBV(ICBVS,3)=NALL(NCOUNT+(NPX+1)*C20)
        WRITE(20,230) ICBVS, SELBVN(ICBVS), ICBVS, SELBV(ICBVS,1), ICBVS,
+   SELBV(ICBVS,2), ICBVS, SELBV(ICBVS,3), I, J, ICB
230  FORMAT('SELBVN(',I3,') = ',I5,' SELBV(',I3,',1) = ',I5,
+   ' SELBV(',I3,',2) = ',I5,' SELBV(',I3,',3) = ',I5,/,
+   'I = ',I5,' J = ',I5,' ICB = ',I5)
235  CONTINUE
240  CONTINUE
      ENDIF
C
      CLOSE(20)
C
      RETURN
      END

```

```

C*****
C
  SUBROUTINE FEMINP(NPX,NPY,NEX,NEY,C20,ICS,ICBH,ICBV,
+NTFLAT,NTOT,PNLSE,ITN,UPRE,T,SXPS,SYPS,XYPS,NALL,
+NCX,NCY,NCZ,ELS,ELSN,ELBHN,ELBH,ELBVN,ELBV)
C
C*****
C      Subroutine to write the ABAQUS input deck
C*****
C
  DOUBLE PRECISION SXPS(301),SYPS(301),XYPS(301),
+NCX(1000),NCY(1000),NCZ(1000),T(301)
  INTEGER ITN,ELS(1000,1000),ELSN(1000),ELBHN(1000),
+ELBH(1000,1000),ELBVN(1000),ELBV(1000,1000),
+NALL(1000),PNLSE(1000,20),C20
  CHARACTER*(3)UPRE
  CHARACTER*(1)COMMA
C
  OPEN(30,FILE='femmodel.inp')
C
  COMMA=', '
  NP=NPX*NPY
  NSEPP=NEX*NEY
  NONX=C20*NPX*NEX+1
  NONY=C20*NPY*NEY+1
C
  WRITE(30,700)
700  FORMAT('*PREPRINT,HISTORY=NO,ECHO=NO,CONTACT=NO',/,
+ '*RESTART,WRITE,FREQ=1')
  IF(ITN.GE.2)THEN
  WRITE(30,705)
705  FORMAT('*INITIAL CONDITIONS,TYPE=STRESS')
  DO 703 I=1,NP
  WRITE(30,704)I,COMMA,SXPS(I),COMMA,SYPS(I),COMMA,
+SXYPS(I),COMMA
704  FORMAT('E-PL',I3,A1,G20.12,A1,G20.12,A1,G20.12,A1)
703  CONTINUE
  ENDIF
C
C      write nodal coordinates (includes eccentricity nodes)
C
  WRITE(30,706)
706  FORMAT('*NODE,NSET=N-ALL')

```

```

      DO 266 NCOUNT=1,NTOT
        WRITE(30,FMT=710)NALL(NCOUNT),COMMA,NCX(NCOUNT),COMMA,
+       NCY(NCOUNT),COMMA,NCZ(NCOUNT)
710     FORMAT(I7,A1,G15.8,A1,G15.8,A1,G15.8)
266     CONTINUE
C
C       generate node sets for BEAM MPC
C
      NINC=0
      WRITE(30,761)
761     FORMAT('*NSET,NSET=N-SUPR')
      DO 764 I=1,NONY,1
        NINC=(I-1)*NONX
      DO 771 NCOUNT=1+NEX*C20+NINC,NONX+NINC,NEX*C20
        WRITE(30,765)NALL(NCOUNT),COMMA
765     FORMAT(I7,A1)
771     CONTINUE
764     CONTINUE
      NINC=0
      WRITE(30,781)
781     FORMAT('*NSET,NSET=N-ECC')
      DO 784 I=1,NONY,1
        NINC=(I-1)*(NPX+1)
      DO 783 NCOUNT=NTFLAT+2+NINC,NTFLAT+NPX+1+NINC,1
        WRITE(30,788)NALL(NCOUNT),COMMA
788     FORMAT(I7,A1)
783     CONTINUE
784     CONTINUE
C
C       write shell elements
C
      WRITE(30,720)
720     FORMAT('*ELEMENT,TYPE=S8R5,ELSET=E-MEMB')
      DO 280 J=1,ICS
        WRITE(30,721)ELSN(J),COMMA,ELS(J,1),COMMA,ELS(J,2),
+       COMMA,ELS(J,3),COMMA,ELS(J,4),COMMA,ELS(J,5),COMMA,
+       ELS(J,6),COMMA,ELS(J,7),COMMA,ELS(J,8)
721     FORMAT(I6,A1,I6,A1,I6,A1,I6,A1,I6,A1,I6,A1,I6,A1,I6,A1,I6)
280     CONTINUE
C
C       generate separate element sets for each panel (shell elements only)
C       separate panel thicknesses
C       - max no of panels: 999
C
      DO 725 I=1,NP

```

```

      IF(I.LE.9)THEN
        WRITE(30,754)I
754   FORMAT('*ELSET,ELSET=E-PL',I1)
      ELSE IF(I.GE.10.AND.I.LE.99)THEN
        WRITE(30,752)I
752   FORMAT('*ELSET,ELSET=E-PL',I2)
      ELSE
        WRITE(30,753)I
753   FORMAT('*ELSET,ELSET=E-PL',I3)
      ENDIF
      DO 727 J=1,NSEPP
        WRITE(30,728)PNLSE(I,J),COMMA
728   FORMAT(I7,A1)
727   CONTINUE
725  CONTINUE
C
C      create beam elements
C
      WRITE(30,730)
730  FORMAT('*ELEMENT,TYPE=B32,ELSET=E-BHORI')
      DO 290 J=1,ICBH
        WRITE(30,FMT=731)ELBHN(J),COMMA,ELBH(J,1),COMMA,
+      ELBH(J,2),COMMA,ELBH(J,3)
731   FORMAT(I6,A1,I6,A1,I6,A1,I6)
290   CONTINUE
      WRITE(30,740)
740  FORMAT('*ELEMENT,TYPE=B32,ELSET=E-BVERT')
      DO 300 J=1,ICBV
        WRITE(30,FMT=741)ELBVN(J),COMMA,ELBV(J,1),COMMA,
+      ELBV(J,2),COMMA,ELBV(J,3)
741   FORMAT(I6,A1,I6,A1,I6,A1,I6)
300   CONTINUE
C
C      create seperate element sets for each upright so that
C      max no of upright sets = 999
C
      BVTOT=0
      NUPRS=ICBV/NEY
      WRITE(10,801)NUPRS
801  FORMAT('NUPRS = ',I3)
      DO 800 I=1,NUPRS
        IF(I.LE.9)THEN
          WRITE(30,810)I
810   FORMAT('*ELSET,ELSET=E-UP',I1)
        ELSE IF(I.GE.10.AND.I.LE.99)THEN

```

```

        WRITE(30,811)I
811    FORMAT(' *ELSET,ELSET=E-UP',I2)
        ELSE
        WRITE(30,812)I
812    FORMAT(' *ELSET,ELSET=E-UP',I3)
        ENDIF
        DO 805 J=1,NEY
            BVTOT=BVTOT+1
            WRITE(30,815)ELBVN(BVTOT),COMMA
815    FORMAT(I6,A1)
805    CONTINUE
800    CONTINUE
C
        WRITE(30,808)
808    FORMAT(' *NSET,NSET=N-ENDC,ELSET=E-UP7')
C
C        create separate element sets for each part of the flange
C        between two uprights; max no of flange sets = 999
C
        BHTOT=0
        NFLS=ICBH/NEX
        WRITE(10,840)NFLS
840    FORMAT('NFLS = ',I3)
        DO 850 I=1,NFLS
            IF(I.LE.9)THEN
                WRITE(30,845)I
845    FORMAT(' *ELSET,ELSET=E-FL',I1)
            ELSE IF(I.GE.10.AND.I.LE.99)THEN
                WRITE(30,846)I
846    FORMAT(' *ELSET,ELSET=E-FL',I2)
            ELSE
                WRITE(30,847)I
847    FORMAT(' *ELSET,ELSET=E-FL',I3)
            ENDIF
            DO 855 J=1,NEX
                BHTOT=BHTOT+1
                WRITE(30,835)ELBHN(BHTOT),COMMA
835    FORMAT(I6,A1)
855    CONTINUE
850    CONTINUE
C
C        create separate element sets for the upper and lower flange
C
        WRITE(30,742)
742    FORMAT(' *ELSET,ELSET=E-BH1')

```

```

DO 743 I=1,NEX*NPX
  WRITE(30,745)ELBHN(I),COMMA
745  FORMAT(I6,A1)
743  CONTINUE
  WRITE(30,746)
746  FORMAT('*ELSET,ELSET=E-BH2')
  IF(NEX*NPX.EQ.1)THEN
    IY=2
  ELSE
    IY=NEX*NPX+1
  ENDIF
  DO 747 J=IY,NEX*NPX*(NPY+1)
    WRITE(30,748)ELBHN(J),COMMA
748  FORMAT(I6,A1)
747  CONTINUE
  WRITE(30,813)
813  FORMAT('*NSET,NSET=N-UPFL,ELSET=E-BH2',/,
+ '*NSET,NSET=N-LOWFL,ELSET=E-BH1')
C
C      write shell section properties for all shells
C
DO 880 I=1,NP
  IF(I.LE.9)THEN
    WRITE(30,882)I,T(I),COMMA
882  FORMAT('*SHELL SECTION,ELSET=E-PL',I1,',MATERIAL=ALU-S',
+/,G12.7,A1)
    ELSE IF(I.GE.10.AND.I.LE.99)THEN
      WRITE(30,884)I,T(I),COMMA
884  FORMAT('*SHELL SECTION,ELSET=E-PL',I2,',MATERIAL=ALU-S',
+/,G12.7,A1)
    ELSE
      WRITE(30,886)I,T(I),COMMA
886  FORMAT('*SHELL SECTION,ELSET=E-PL',I3,',MATERIAL=ALU-S',
+/,G12.7,A1)
    ENDIF
880  CONTINUE
C
  WRITE(30,755)
755  FORMAT('*BEAM SECTION,ELSET=E-BH1,MATERIAL=ALU-B,SECTION=I',/,
+ '0.0055,0.02937,0.0571,0.0,0.00318,0.0,0.00238',/,
+ '0.0,0.0,-1.0',/,
+ '*BEAM SECTION,ELSET=E-BH2,MATERIAL=ALU-B,SECTION=I',/,
+ '0.02952,0.0381,0.0,0.05874,0.0,0.00519,0.00397',/,
+ '0.0,0.0,-1.0',/,
+ '*BEAM SECTION,ELSET=E-BVERT,MATERIAL=ALU-B,SECTION=L',/,

```



```

+ '0.0254,0.0254,0.003175,0.003175' ,/,
+ '0.0,0.0,-1.0' ,/,
+ '*MATERIAL,NAME=ALU-B' ,/, '*ELASTIC' ,/, '71.0E9,0.3' ,/,
+ '*MATERIAL,NAME=ALU-S' ,/, '*ELASTIC' ,/, '72.4E9,0.3' ,/,
+ '*BOUNDARY' ,/, '1,1,6' ,/, '38,1,6' ,/, '75,1,6' ,/,
+ '112,1,6' ,/, '149,1,6' ,/, '186,1,6' ,/, '223,1,6')
C
  IF (UPRE.EQ. 'ECC') THEN
    WRITE(30,FMT=759)
759   FORMAT(' *MPC' ,/, 'BEAM,N-SUPR,N-ECC')
    ENDIF
C
  WRITE(30,760)
760   FORMAT(' *STEP' ,/, '*STATIC')
C
  WRITE(30,FMT=762)
C
762   FORMAT(' *LOAD' ,/, '37,2,8578.3' ,/, '74,2,8578.3' ,/,
+ '111,2,8578.3' ,/, '148,2,8578.3' ,/, '185,2,8578.3' ,/,
+ '222,2,8578.3' ,/, '259,2,8578.3')
C
  WRITE(30,FMT=770)
770   FORMAT(' *EL FILE' ,/, 'S,SINV' ,/, '*NODE FILE' ,/, 'COORD')
  WRITE(30,FMT=780)
780   FORMAT(' *EL PRINT,TOTALS=YES' ,/, 'S,MISES' ,/, 'SP' ,/, '*END STEP')
  ENDFILE(UNIT=30)
C
  CLOSE(30)
C
  RETURN
  END

```