

Appendix E

Code

E.1 Subroutines for the isotropic 8β element

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C=====SHL8
C
C   ELEMENT STIFFNESS MATRIX FOR AN ASSUMED STRESS
C   FOUR NODE MEMBRANE ELEMENT WITH DRILLING DEGREES OF
C   FREEDOM - 8 BETA PARAMETERS
C
C   (K+P)a = f - DISPLACEMENT FORMULATION
C   (K+h)a = f - MIXED FORMULATION
C
C-----
SUBROUTINE SHL8(XYZ, ID, MAXN, NSHL8)

  IMPLICIT REAL*8 (A-H, O-Z)
  PARAMETER (NP=8)

  DIMENSION XYZ(4, MAXN), ID(6, MAXN), S(24, 24)
  DIMENSION RS(2, 16), GWS(2, 16), B(3, 14), BY(1, 14)
  DIMENSION T1(3, 14), T4(14, 14), T1Y(1, 14), T2Y(14, 14)
  DIMENSION BTAY(3, 4), BTAY2(3, 4)
  DIMENSION P(3, NP), P1(3, NP), T2(NP, 14), T3(NP, 14)
  DIMENSION P2(NP, NP), XDU(NP, NP), CINV(3, 3)
  DIMENSION NDOF(6), PPURE(3, NP)
  DIMENSION PTRANSO(3, 3), PTRANS(3, 3), QTRANS(3, 3)
  DIMENSION IPVT(NP), WORK(NP), TEST1(NP, NP)
  DIMENSION NEIG(24), SEIG(24, 24)
  DIMENSION BPL(3, 12), BS(2, 12), T1S(2, 12)
  DIMENSION T1PL(6, 12), T2PL(12, 12)
  DIMENSION DB(3, 3), DS(2, 2)

  COMMON /CONSTR/ ICONSTR(9)

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APPENDIX E. CODE

```
COMMON /VECT / V(4,3),XYL(3,5)
COMMON /GENR / NAM,INE,INI,NIDENT
COMMON /SIDE / ALPHA(4),XLENG(4)
COMMON /TEMP / IDE,II,JJ,KK,LL,HH,EE,UU
COMMON /WEIGHI/ WGHTO
COMMON /IOLIST/ NTM,NTR,NIN,NOT,NSP,NFL,NT7,NT8

DATA ZERO /0.DO/, ONE /1.DO/, TWO /2.DO/, FOUR /4.DO/
DATA NEIG /1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,
&          19,20,21,22,23,24/
DATA NDOF /1,2,3,4,5,6/
DATA FF /1.2D0/

IAXIS=1

ND=24
NUMN=4
NTOT=0
NTELE=2          ! DEFAULT - DISPLACEMENT FORMULATION
INT=9           ! DEFAULT - FULL INTEGRATION
INTTAY=4
INT2=4
NWARP=0         ! DEFAULT - WARP CORRECTION NOT ACTIVE
NPRINT=0
NSTAT=1
NEIGEN=0
NLOCK=1         ! DEFAULT - LOCKING CORRECTION ACTIVE
WGHTO=0.01
GG=-1.         ! DEFAULT - GG=EE/(TWO+TWO*UU)
IMIXED=5       ! DEFAULT - NT-FORMULATION
ALPHACORR=ONE
IPSTATE=2      ! DEFAULT - P-MATRIX CLASSIFICATION

C-----LOOP THROUGH ALL ELEMENTS-----
DO 400 MM=1,NSHL8

C-----READ NODE NUMBERS AND MATERIAL PROPERTIES-----
C-----AND ELEMENT PARAMETERS-----
NAM=0
NIDENT=0
CALL CFREE
CALL CFREEPT
CALL CFREEI(' ',IDE,1)
CALL CFREEI('N',II,4)
CALL CFREER('H',HH,1)
CALL CFREER('E',EE,1)
CALL CFREER('U',UU,1)
CALL CFREEI('G',NAM,4)
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```
CALL CFREEI('I',INT,1)
CALL CFREEI('J',INTTAY,1)
CALL CFREEI('K',INT2,1)
CALL CFREER('X',GG,1)
CALL CFREEI('S',NSTAT,1)
CALL CFREEI('T',NTELE,1)
CALL CFREEI('B',IPSTATE,1)
CALL CFREEI('L',NEIGEN,1)
CALL CFREEI('Z',NLOCK,1)
CALL CFREER('D',WGHTO,1)
CALL CFREEI('W',NWARP,1)
CALL CFREEI('A',IMIXED,1)
CALL CFREER('Y',ALPHACORR,1)
CALL CFREEI('F',ICONSTR,9)      ! READ DIFFERENT P-MATRIX

IF (GG.LT.ZERO) GG=EE/(TWO+TWO*UU)
IF (GG.EQ.ZERO) NTELE=3
IF (NAM.EQ.0) NIDENT=0
NCHCK=NAM

C-----PRINT ELEMENT DATA-----
10  IF (NPRINT.NE.0) GO TO 15
    CALL TOP
    WRITE (NOT,2003) IMIXED,INT
    WRITE (NOT,2000)
15  WRITE (NOT,2001) IDE,II,JJ,KK,LL,HH,EE,GG,UU
    NPRINT=NPRINT+1
    IF(NPRINT.GT.50) NPRINT=0

C-----SKIP FOR IDENTICAL ELEMENTS-----
IF (NIDENT.LT.0.OR.NIDENT.GT.1) NIDENT=0
IF (NIDENT.EQ.1.AND.NCHCK.GT.NAM) GOTO 295

C-----LOCAL COORDINATES, SIDE LENGTHS & ANGLES-----
CALL LOCAL(XYZ,MAXN,IAXIS)

DO 19 I=1,4
CALL ANGLN(I,ALPHA(I),XLENG(I))
19  CONTINUE

C-----
C
C ELEMENT K MATRIX - MIXED AND DISPL. FORMULATION
C
C-----

C-----CALCULATE COMPLIANCE MATRIX-----
CALL MATLWINV(NSTAT,EE,UU,CINV)
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```
C-----INITIALIZE MATRICES-----
      DO 20 I=1,24
      DO 20 J=1,24
20    S(I,J)=ZERO

      DO 22 J=1,14
      B(1,J)=ZERO
      B(2,J)=ZERO
      B(3,J)=ZERO
      T1Y(1,J)=ZERO
      BY(1,J)=ZERO
22    CONTINUE

      DO 23 I=1,NP
      DO 23 J=1,NP
23    P2(I,J)=ZERO

      DO 24 I=1,14
      DO 24 J=1,14
      T2Y(I,J)=ZERO
24    CONTINUE

      DO 25 I=1,NP
      DO 25 J=1,14
25    T2(I,J)=ZERO

      DO 26 J=1,4
      BTAY(1,J)=ZERO
      BTAY(2,J)=ZERO
      BTAY(3,J)=ZERO
26    CONTINUE

      DO 27 I=1,3
      DO 27 J=1,NP
27    P(I,J)=ZERO

      CALL FORMTRANSC (PTRANSO)
      CALL MEMXB3 (BY,DETJO)

C-----CALCULATE ELEMENT AREA-----
      CALL ELAREA(AREA)

C-----CALCULATE MEMBRANE LOCKING CORRECTION-----
      IF (NLOCK.EQ.1) THEN
      CALL INTPTS(INTTAY,RS,GWS)
      DO 35 I=1,INTTAY
      CALL MEMLOK(I,RS,BTAY2,DETJ)
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DO 35 K=1,3
DO 35 L=1,4
BTAY(K,L)=BTAY(K,L)+BTAY2(K,L)*GWS(1,I)*GWS(2,I)*DETJ/AREA
35 CONTINUE
ENDIF

C-----DEFINE INTEGRATION POINTS-----
CALL INTPTS(INT,RS,GWS)

C-----LOOP THROUGH THE INTEGRATION POINTS-----
DO 80 I=1,INT
CALL MEMXBTJ (I,RS,B,DETJ,BTAY,NLOCK,AJM11,AJM12,AJM21,AJM22)
IF (IPSTATE.EQ.2) THEN
CALL HQ8P2 (I,RS,PPURE) ! DETERMINE P
ELSEIF (IPSTATE.EQ.1) THEN
CALL HQ8P1 (I,RS,PPURE) ! DETERMINE P
ELSE
CALL HQ8PA (I,RS,PPURE) ! DETERMINE P
ENDIF

CALL FIXP (P,PTRANSO,PTRANS,QTRANS,AJM11,AJM12,AJM21,AJM22,
& DETJ,DETJO,PPURE,IMIXED,3,NP) ! TRANSFORM P

C-----EVALUATE INTEGRAL-----
C....C^-1 x P .....
DO 50 K=1,3
DO 50 L=1,NP
C1=CINV(K,1)*P(1,L)+CINV(K,2)*P(2,L)+CINV(K,3)*P(3,L)
P1(K,L)=C1*DETJ*HH*GWS(1,I)*GWS(2,I)
50 CONTINUE

C....P^t x C^-1 x P.....
DO 54 K=1,NP
DO 54 L=1,NP
C1=P(1,K)*P1(1,L)+P(2,K)*P1(2,L)+P(3,K)*P1(3,L)
P2(K,L)=P2(K,L)+C1
54 CONTINUE

C....P^t x B.....
DO 60 K=1,NP
DO 60 L=1,12
C1=P(1,K)*B(1,L)+P(2,K)*B(2,L)+P(3,K)*B(3,L)
T2(K,L)=T2(K,L)+C1*DETJ*HH*GWS(1,I)*GWS(2,I)
60 CONTINUE

C-----END OF INTEGRATION LOOP-----

80 CONTINUE
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C-----DETERMINE INVERSE OF H -----
  CALL XEQY(XDU,P2,NP,NP)
  CALL DGEFA(P2,NP,NP,IPVT,INFO)
  JOB=01
  CALL DGED1(P2,NP,NP,IPVT,DETX,WORK,JOB)
  CALL TESTINVERSE(XDU,P2,NP,NP,TEST1)

C.....H^-1 * G.....
  DO 90 K=1,NP
  DO 90 L=1,12
  C1=P2(K,1)*T2(1,L)+P2(K,2)*T2(2,L)+P2(K,3)*T2(3,L)
  & +P2(K,4)*T2(4,L)+P2(K,5)*T2(5,L)+P2(K,6)*T2(6,L)
  & +P2(K,7)*T2(7,L)+P2(K,8)*T2(8,L)
  T3(K,L)=C1
90  CONTINUE

C.....K = G^t * H^-1 * G.....
  DO 95 K=1,12
  DO 95 L=1,12
  C1=T2(1,K)*T3(1,L)+T2(2,K)*T3(2,L)+T2(3,K)*T3(3,L)
  & +T2(4,K)*T3(4,L)+T2(5,K)*T3(5,L)+T2(6,K)*T3(6,L)
  & +T2(7,K)*T3(7,L)+T2(8,K)*T3(8,L)
  T4(K,L)=C1
95  CONTINUE

C-----
C
C ELEMENT P MATRIX - DISPLACEMENT FORMULATION
C
C-----

      IF (NTELE.EQ.3) GOTO 280
      IF (NTELE.EQ.1) GOTO 200

C-----EVALUATE INTEGRAL AND SUM K AND P MATRICES-----
  DO 140 L=1,12
  T1Y(1,L)=BY(1,L)*DETJO*HH*FOUR
140  CONTINUE

      DO 141 K=1,12
      DO 141 L=1,12
      T2Y(K,L)=BY(1,K)*T1Y(1,L)
141  CONTINUE

      DO 150 K=1,12
      DO 150 L=1,12
      T4(K,L)=T4(K,L)+t2y(k,l)*GG

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150 CONTINUE
    GOTO 280

C-----
C
C ELEMENT h MATRIX - MIXED FORMULATION
C
C-----

C-----DEFINE INTEGRATION POINTS-----
200 CALL INTPTS(INT2,RS,GWS)

C-----LOOP THROUGH THE INTEGRATION POINTS-----
    DO 250 I=1,INT2
      CALL MEMXB2 (I,RS,BY,DETJ)

C-----EVALUATE INTEGRAL-----
    DO 250 L=1,12
      T1Y(1,L)=T1Y(1,L)+BY(1,L)*DETJ*HH*GWS(1,I)*GWS(2,I)
250 CONTINUE

C.....h * hT.....
    DO 260 I=1,12
      DO 260 J=1,12
        T2Y(I,J)=T1Y(1,I)*T1Y(1,J)
260 CONTINUE

C.....K + h.....

      VOLEL=HH*AREA
      DO 270 I=1,12
        DO 270 J=1,12
          T4(I,J)=T4(I,J)+T2Y(I,J)*GG/VOLEL
270 CONTINUE

C-----SUM TO FORM MEMBRANE STIFFNESS-----
280 DO 290 K=1,4
      DO 290 L=1,4
        S(6*K-5,6*L-5)=T4(3*K-2,3*L-2)
        S(6*K-4,6*L-4)=T4(3*K-1,3*L-1)
        S(6*K-5,6*L-4)=T4(3*K-2,3*L-1)
        S(6*K-4,6*L-5)=T4(3*K-1,3*L-2)

C.....DRILLING ROTATIONS.....
      S(6*K-0,6*L-0)=T4(3*K-0,3*L-0)
      S(6*K-5,6*L-0)=T4(3*K-2,3*L-0)
      S(6*K-4,6*L-0)=T4(3*K-1,3*L-0)
      S(6*K-0,6*L-5)=T4(3*K-0,3*L-2)

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S(6*K-0,6*L-4)=T4(3*K-0,3*L-1)
290 CONTINUE

C-----
C
C ELEMENT K MATRIX (BENDING STIFFNESS)
C
C-----

      INT3=4

C-----DEFINE INTEGRATION POINTS-----
      CALL INTPTS(INT3,RS,GWS)

C-----CALCULATE MATERIAL MATRIX-----
      CALL MATPL(EE,UU,HH,FF,DB,DS)

C-----INITIALIZE MATRICES-----
      DO 530 I=1,3
      DO 530 J=1,12
530 BPL(I,J)=ZERO

      DO 531 I=1,2
      DO 531 J=1,12
531 BS(I,J)=ZERO

      DO 535 I=1,12
      DO 535 J=1,12
      T2PL(I,J)=ZERO
535 CONTINUE

C-----CALC. JACOBIAN MATRIX AT SAMP. POINTS-----
      CALL JASAPM

C-----LOOP THROUGH THE INTEGRATION POINTS-----
      DO 590 I=1,INT3
      CALL CPT1B (I,RS,BPL,BS,DETJ)

C-----EVALUATE INTEGRAL (MATRIX MULTIPLICATION)-----
      DO 550 K=1,3
      DO 550 L=1,12
      C1=DB(K,1)*BPL(1,L)+DB(K,2)*BPL(2,L)+DB(K,3)*BPL(3,L)
      T1PL(K,L)=C1*GWS(1,I)*GWS(2,I)*DETJ
550 CONTINUE

      DO 555 K=1,2
      DO 555 L=1,12
      C1=DS(K,1)*BS(1,L)+DS(K,2)*BS(2,L)
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T1S(K,L)=C1*GWS(1,I)*GWS(2,I)*DETJ
555 CONTINUE

DO 560 K=1,12
DO 560 L=1,12
C1=BPL(1,K)*T1PL(1,L)+BPL(2,K)*T1PL(2,L)+BPL(3,K)*T1PL(3,L)
C2=BS (1,K)*T1S (1,L)+BS (2,K)*T1S (2,L)
T2PL(K,L)=T2PL(K,L)+C1+C2
560 CONTINUE

590 CONTINUE

C-----END INTEGRATION LOOP-----

C-----SUM TO FORM ELEMENT STIFFNESS MATRIX-----
DO 580 K=1,4
DO 580 L=1,4
S(6*K-3,6*L-3)=T2PL(3*K-2,3*L-2)
S(6*K-2,6*L-2)=T2PL(3*K-1,3*L-1)
S(6*K-1,6*L-1)=T2PL(3*K-0,3*L-0)
S(6*K-3,6*L-2)=T2PL(3*K-2,3*L-1)
S(6*K-2,6*L-3)=T2PL(3*K-1,3*L-2)
S(6*K-3,6*L-1)=T2PL(3*K-2,3*L-0)
S(6*K-1,6*L-3)=T2PL(3*K-0,3*L-2)
S(6*K-2,6*L-1)=T2PL(3*K-1,3*L-0)
S(6*K-1,6*L-2)=T2PL(3*K-0,3*L-1)
580 CONTINUE

C-----CONVERT FROM FLAT TO WARPED-----
IF (NWARP.NE.0) CALL TWISTM(S)

C-----ROTATE STIFNESS TO GLOBAL SYSTEM-----
CALL ROTATE(S)

C-----IDENTIFY DOF WHICH HAVE STIFFNESS-----
295 DO 320 K=1,6
IF (S(K,K).NE.ZERO) ID(K,II)=1
IF (S(K+6,K+6).NE.ZERO) ID(K,JJ)=1
IF (S(K+12,K+12).NE.ZERO) ID(K,KK)=1
IF (S(K+18,K+18).NE.ZERO) ID(K,LL)=1
320 CONTINUE

C-----SAVE ELEMENT ARRAYS-----
WRITE (NT7) IDE,NUMN,II,JJ,KK,LL,NDOF,ND
WRITE (NT8) S
WRITE (NSP) IDE,II,JJ,KK,LL,XYL,T3,IMIXED,PTRANSO,DETJO,
& BTAY,NLOCK,ALPHACORR,NWARP

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C-----SAVE STIFFNESS MATRIX TO SOLVE EIGENVALUES-----
  IF (NEIGEN.NE.0) THEN
    NPRINT=0
    NEX=24
    DO 350 I=1,NEX
    DO 350 J=1,NEX
350  SEIG(I,J)=S(NEIG(I),NEIG(J))
    CALL CALJAC(SEIG,NEX,NEIGEN)
    ENDIF

C-----CHECK FOR ELEMENT GENERATION-----
  IF (NAM.EQ.0) GOTO 400
  NAM=NAM-1
  IDE=IDE+INE
  II=II+INI
  JJ=JJ+INI
  KK=KK+INI
  LL=LL+INI
  NTOT=NTOT+1
  GOTO 10
400  IF ((MM+NTOT).EQ.NSHL8) RETURN

C-----
2000 FORMAT (/,' EL. #   I   J   K   L           H           E'
*           ,'           G   U ')
2001 FORMAT (1I6,4I5,1F9.3,2E12.3,1F9.3)
2002 FORMAT (5E10.3)
2003 FORMAT (/,' ELEMENT TYPE: SHL8, Version: ',1I1,/,
*           ' INTEGRATION SCHEME: ',1I2,'-POINT')
  END

C-----

C=====MEMXBTJ
C
C  SUBROUTINE EVALUATES THE OPERATOR MATRIX
C  [B-MATRIX] FOR THE ELEMENT K MATRIX OF A
C  FOUR NODE 8 DOF MEMBRANE ELEMENT WITH IN-
C  PLANE DRILLING ROTATIONS.
C
C-----
  SUBROUTINE MEMXBTJ(I,RS,B,DETJ,BTAY,NLOCK,
&                  AJM11,AJM12,AJM21,AJM22)

  IMPLICIT REAL*8 (A-H,O-Z)

  DIMENSION RS(2,16),QR(9),QS(9),B(3,14),BTAY(3,4)

  COMMON /VECT / V(4,3),XYL(3,5)

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COMMON /TEMP / IDE,II,JJ,KK,LL,HH,EE,UU
COMMON /SIDE / ALPHA(4),XLENG(4)
COMMON /IOLIST/ NTM,NTR,NIN,NOT,NSP,NFL,NT7,NT8

DATA ZERO /0.DO/, ONE /1.DO/, TWO /2.DO/,
&      FOUR /4.DO/, EIGHT /8.DO/

RCOORD=RS(1,I)
SCCOORD=RS(2,I)

XYL11=XYL(1,1)
XYL21=XYL(2,1)
XYL12=XYL(1,2)
XYL22=XYL(2,2)
XYL13=XYL(1,3)
XYL23=XYL(2,3)
XYL14=XYL(1,4)
XYL24=XYL(2,4)

C-----CALCULATE SHAPE FCN. DERIVITIVES AT GIVEN POINT-----
C.....CALCULATE LINEAR FUNCTIONS.....
QR1=-(ONE-SCCOORD)/FOUR
QR2= (ONE-SCCOORD)/FOUR
QR3= (ONE+SCCOORD)/FOUR
QR4=-(ONE+SCCOORD)/FOUR
QR(1)=QR1
QR(2)=QR2
QR(3)=QR3
QR(4)=QR4

C.....CALCULATE QUADRATIC FUNCTIONS.....
QR(5)=-RCOORD*(ONE-SCCOORD)
QR(6)= (ONE-SCCOORD)*(ONE+SCCOORD)/TWO
QR(7)=-RCOORD*(ONE+SCCOORD)
QR(8)=- (ONE-SCCOORD)*(ONE+SCCOORD)/TWO
QR(9)=-TWO*RCOORD*(ONE-SCCOORD**TWO)

C.....CALCULATE LINEAR FUNCTIONS.....
QS1=-(ONE-RCOORD)/FOUR
QS2=-(ONE+RCOORD)/FOUR
QS3= (ONE+RCOORD)/FOUR
QS4= (ONE-RCOORD)/FOUR
QS(1)=QS1
QS(2)=QS2
QS(3)=QS3
QS(4)=QS4

C.....CALCULATE QUADRATIC FUNCTIONS.....

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```

QS(5)=- (ONE+RCOORD)*(ONE-RCOORD)/TWO
QS(6)=-SCOORD*(ONE+RCOORD)
QS(7)= (ONE+RCOORD)*(ONE-RCOORD)/TWO
QS(8)=-SCOORD*(ONE-RCOORD)
QS(9)=-TWO*SCOORD*(ONE-RCOORD**TWO)

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C-----CALCULATE JACOBIAN MATRIX-----

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AJM11=QR1*XYL11+QR2*XYL12 + QR3*XYL13+QR4*XYL14
AJM12=QR1*XYL21+QR2*XYL22 + QR3*XYL23+QR4*XYL24
AJM21=QS1*XYL11+QS2*XYL12 + QS3*XYL13+QS4*XYL14
AJM22=QS1*XYL21+QS2*XYL22 + QS3*XYL23+QS4*XYL24

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C-----INVERT JACOBIAN MATRIX-----

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DETJ=AJM11*AJM22-AJM12*AJM21
IF (DETJ.LE.0.) WRITE(NOT,2000)DETJ
IF (DETJ.LE.0.) WRITE(NTM,2000)DETJ
AJI11= AJM22/DETJ
AJI22= AJM11/DETJ
AJI12=-AJM12/DETJ
AJI21=-AJM21/DETJ

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C-----CALCULATE SIDE LENGTHS AND ANGLES-----

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DO 40 IC=1,4
MC=IC+4
TEMP=ONE/IC
LC=MC-1+4*INT(TEMP)
KC=MOD(MC,4)+1
JC=LC-4
XLENIJ=XLENG(JC)
XLENIK=XLENG(IC)
ALPHIJ=ALPHA(JC)
ALPHIK=ALPHA(IC)

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C-----EVALUATE B-MATRIX-----

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B(1,3*IC-2)=QR(IC)*AJI11+QS(IC)*AJI12
B(3,3*IC-1)=B(1,3*IC-2)
B(2,3*IC-1)=QR(IC)*AJI21+QS(IC)*AJI22
B(3,3*IC-2)=B(2,3*IC-1)
B(1,3*IC)=XLENIJ*DCOS(ALPHIJ)/EIGHT*
& (QR(LC)*AJI11+QS(LC)*AJI12)
& -XLENIK*DCOS(ALPHIK)/EIGHT*
& (QR(MC)*AJI11+QS(MC)*AJI12)
B(2,3*IC)=XLENIJ*DSIN(ALPHIJ)/EIGHT*
& (QS(LC)*AJI22+QR(LC)*AJI21)
& -XLENIK*DSIN(ALPHIK)/EIGHT*
& (QS(MC)*AJI22+QR(MC)*AJI21)
B(3,3*IC)=XLENIJ*DCOS(ALPHIJ)/EIGHT*
& (QS(LC)*AJI22+QR(LC)*AJI21)

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&      -XLENIK*DCOS(ALPHIK)/EIGHT*
&      (QS(MC)*AJI22+QR(MC)*AJI21)
&      +XLENIJ*DSIN(ALPHIJ)/EIGHT*
&      (QR(LC)*AJI11+QS(LC)*AJI12)
&      -XLENIK*DSIN(ALPHIK)/EIGHT*
&      (QR(MC)*AJI11+QS(MC)*AJI12)

C-----CORRECT SHAPE FUNCTIONS: MEMBRANE LOCKING-----
      IF (NLOCK.EQ.1) THEN
      B(1,3*IC)=B(1,3*IC)-BTAY(1,IC)
      B(2,3*IC)=B(2,3*IC)-BTAY(2,IC)
      B(3,3*IC)=B(3,3*IC)-BTAY(3,IC)
      ENDIF
40  CONTINUE

C-----ADD BUBBLE-----
      B(1,13)=QR(9)*AJI11+QS(9)*AJI12
      B(2,14)=QR(9)*AJI21+QS(9)*AJI22
      B(3,13)=B(2,14)
      B(3,14)=B(1,13)
      RETURN

C-----
2000 FORMAT(' JACOBIAN=',E15.6,
*          ' NODE NUMBERS NOT IN ORDER ')
      END

C-----
C=====HQ8PA
C
C  SUBROUTINE EVALUATES THE P MATRIX OF A HYBRID FOUR NODE
C  MEMBRANE ELEMENT USING 8 BETA PARAMETERS.
C
C-----
      SUBROUTINE HQ8PA(I,RS,P)

      IMPLICIT REAL*8 (A-H,O-Z)
      PARAMETER (NPOSS=57)

      DIMENSION RS(2,16),P(3,8), SM(3,NPOSS)

      COMMON /IOLIST/ NTM,NTR,NIN,NOT,NSP,NFL,NT7,NT8
      COMMON /CONSTR/ ICONSTR(9)

      DATA ZERO /0.DO/, ONE /1.DO/, TWO /2.DO/

C-----OBTAIN R & S COORDINATES-----
      R=RS(1,I)

```



APPENDIX E. CODE

```
S=RS(2,I)

C-----INITIALIZE-----
DO 10 K=1,3
DO 10 L=1,8
10 P(K,L)=ZERO
DO 20 K=1,3
DO 20 L=1,NPOSS
20 SM(K,L)=ZERO

C-----EVALUATE P-----
SM(1, 1)=ONE
SM(2, 2)=ONE
SM(3, 3)=ONE

SM(1, 4)=R
SM(1, 5)=S
SM(2, 6)=R
SM(2, 7)=S
SM(3, 8)=R
SM(3, 9)=S

SM(1,10)=ONE
SM(2,10)=ONE
SM(1,11)=ONE
SM(2,11)=-ONE

SM(2,12)=-S
SM(3,12)=R
SM(1,13)=-R
SM(3,13)=S

SM(1,14)=R*R
SM(2,15)=R*R
SM(3,16)=R*R
SM(1,17)=S*S
SM(2,18)=S*S
SM(3,19)=S*S

SM(1,20)=R*S
SM(2,21)=R*S
SM(3,22)=R*S

SM(1,23)=-R*R
SM(2,24)=-R*R
SM(3,25)=-R*R
SM(1,26)=-S*S
SM(2,27)=-S*S
```




APPENDIX E. CODE

SM(3,28)=-S*S

SM(1,29)=-R*S

SM(2,30)=-R*S

SM(3,31)=-R*S

SM(1,32)=S*S

SM(2,32)=-R*R

SM(1,33)=S

SM(2,33)=-R

SM(1,34)=S

SM(3,34)=-R

SM(1,35)=-R

SM(2,35)=S

SM(2,36)=S

SM(3,36)=-R

SM(1,37)=-R

SM(3,37)=S

SM(2,38)=-R

SM(3,38)=S

SM(1,39)=R

SM(2,39)=-S

SM(1,40)=R

SM(3,40)=-S

SM(1,41)=-S

SM(2,41)=R

SM(2,42)=R

SM(3,42)=-S

SM(1,43)=-S

SM(3,43)=R

SM(2,44)=-S

SM(3,44)=R

SM(1,45)=S*S

SM(3,45)=-R*R

SM(1,46)=-R*R

SM(2,46)=S*S

SM(2,47)=S*S

SM(3,47)=-R*R

SM(1,48)=-R*R

SM(3,48)=S*S

SM(2,49)=-R*R

SM(3,49)=S*S



```

SM(1,50)=R*R
SM(2,50)=-S*S
SM(1,51)=R*R
SM(3,51)=-S*S
SM(1,52)=-S*S
SM(2,52)=R*R
SM(2,53)=R*R
SM(3,53)=-S*S
SM(1,54)=-S*S
SM(3,54)=R*R
SM(2,55)=-S*S
SM(3,55)=R*R

SM(1,56)=S*S
SM(3,56)=-R*S
SM(2,57)=-R*R
SM(3,57)=R*S

DO 50 K=1,8
DO 40 L=1,3
P(L,K)=SM(L,ICONSTR(K))
40 CONTINUE
50 CONTINUE
RETURN

END

C-----
C=====FIXP
C
C A-MATRIX FOR AN ASSUMED STRESS FOUR NODE MEMBRANE ELEMENT
C WITH DRILLING DEGREES OF FREEDOM - 8 BETA PARAMETERS
C
C-----
SUBROUTINE FIXP (P,PTRANS0,PTRANS,QTRANS,AJM11,AJM12,
& AJM21,AJM22,DETJ,DETJO,PPURE,IMIXED,I,J)

IMPLICIT REAL*8 (A-H,O-Z)

DIMENSION P(I,J), PTRANS0(3,3), PTRANS(3,3), QTRANS(3,3)
DIMENSION PPURE(I,J)

DATA ZERO /0.DO/, ONE /1.DO/, TWO /2.DO/

GOTO (10,20,30,40,50,60) IMIXED
STOP 'ILLEGAL MIXED ELEMENT FORMULATION SPECIFIED'

C-----NC (OR I) NOT CONSTRAINED-----

```

```

10  DO 15 K=1,I
    DO 15 L=4,J
    P(K,L)=PPURE(K,L)
15  CONTINUE
    GOTO 500

C-----EP EQUILIBRIUM CONSTRAINED-----
20  DO 25 K=1,I
    DO 25 L=4,J
    C1=PTRANSO(K,1)*PPURE(1,L)+PTRANSO(K,2)*PPURE(2,L)+
    &  PTRANSO(K,3)*PPURE(3,L)
    P(K,L)=C1
25  CONTINUE
    GOTO 500

C-----OC ORTHOGONAL CONSTRAINED-----
30  DO 35 K=1,I
    DO 35 L=4,J
    C1=PTRANSO(K,1)*PPURE(1,L)+PTRANSO(K,2)*PPURE(2,L)+
    &  PTRANSO(K,3)*PPURE(3,L)
    P(K,L)=C1/(DETJ)
35  CONTINUE
    GOTO 500

C-----EC ELEMENT OPTIMALLY CONSTRAINED-----
40  STOP ' 4: NOT IMPLEMENTED '
    GOTO 500

C-----NT NORMALIZED TRANSFORMATION OF STRESS FIELD-----
50  DO 55 K=1,I
    DO 55 L=4,J
    CALL FORMTRANSJ (PTRANS,AJM11,AJM12,AJM21,AJM22)
    C2=PTRANS (K,1)*PPURE(1,L)+PTRANS (K,2)*PPURE(2,L)+
    &  PTRANS (K,3)*PPURE(3,L)
    P(K,L)=C2/(DETJ**2)
55  CONTINUE
    GOTO 500

C-----PH PHYSICAL COMPONENTS IN ISOPARAMETRIC SPACE-----
60  DO 65 K=1,I
    DO 65 L=4,J
    CALL FORMTRANSJQ (QTRANS,AJM11,AJM12,AJM21,AJM22,DETJ)
    C3=QTRANS (K,1)*PPURE(1,L)+QTRANS (K,2)*PPURE(2,L)+
    &  QTRANS (K,3)*PPURE(3,L)
    P(K,L)=C3
65  CONTINUE
    GOTO 500

```



APPENDIX E. CODE

```
C-----ASSIGN LOWER ORDER STRESS-----
500 DO 550 K=1,3
      DO 550 L=1,3
      P(K,L) = PPURE(K,L)
550 CONTINUE
      RETURN

      END

C-----
C=====MATLWINV
C
C INVERSE MATERIAL LAW FOR MEMBRANE ELEMENTS
C COMPLIANCE MATRIX
C-----
      SUBROUTINE MATLWINV(NSTAT,EE,UU,C)

      IMPLICIT REAL*8 (A-H,O-Z)

      DIMENSION C(3,3)

      DATA ZERO /0.DO/, ONE /1.DO/, TWO /2.DO/, HALF /0.50DO/

C-----MEMBRANE PLANE STRESS-----
      IF (NSTAT.EQ.2) GO TO 20
      ENTRY=ONE/EE
      C(1,1)=ENTRY
      C(2,2)=ENTRY
      C(1,2)=-UU/EE
      C(2,1)=-UU/EE
      C(3,3)=TWO*(ONE+UU)/EE
      GO TO 40

C-----MEMBRANE PLANE STRAIN-----
20 ENTRY=( (ONE+UU)*(ONE-TWO*UU) ) / ( TWO * EE * (UU-HALF) )
      C(1,1)=-ENTRY*(ONE-UU)
      C(2,2)=-ENTRY*(ONE-UU)
      C(1,2)= ENTRY*UU
      C(2,1)= ENTRY*UU
      C(3,3)=-ENTRY*TWO

40 C(1,3)=ZERO
      C(2,3)=ZERO
      C(3,1)=ZERO
      C(3,2)=ZERO
      RETURN
```



END

C-----

Appendix F

List of definitions

Advanced elements

Finite elements with both drilling degrees of freedom and an assumed stress interpolation field.

Composite materials

Generally a composite material is a material with two or more constituents, combined by physical process on the macroscopic scale.

Dirichlet problem [63]

Boundary value problem:

$$\begin{aligned}\frac{\partial^2 u}{\partial x_1^2} + \frac{\partial^2 u}{\partial x_2^2} &= 0 && \in R \\ u(x_1, x_2) &= f(x_1, x_2) && \forall (x_1, x_2) \in C\end{aligned}\tag{F.1}$$

Finding a solution of Laplace's equation in a region R with given boundary values, is called a Dirichlet problem. It is known that, if the boundary curve C and the boundary value function are reasonably well behaved, then there exists a unique solution to the Dirichlet problem in (F.1).

Drilling degrees of freedom

In-plane rotational degrees of freedom of membrane elements.

Finite elements [64]

A geometrically complex domain of a problem is represented as a collection of geometrically simple subdomains, called finite elements.

Interpolation functions [64]

Over each finite element the approximation functions are derived using the basic idea that any continuous function can be represented by a linear combination of algebraic polynomials. The approximation functions are derived using concepts from interpolation theory, and are therefore called interpolation functions.

Low order elements

Finite elements with the least possible number of nodes per element. For example a quadrilateral element with only four nodes.

Orthotropic materials

Materials that possess different stiffnesses along three perpendicular axes.

Strain-displacement relations [65]

In small displacement theory the strain-displacement relations are given as follows:

$$\begin{aligned}
 \epsilon_{11} &= \frac{\partial u_1}{\partial x_1} \\
 \epsilon_{22} &= \frac{\partial u_2}{\partial x_2} \\
 \epsilon_{33} &= \frac{\partial u_3}{\partial x_3} \\
 \gamma_{12} &= \frac{\partial u_2}{\partial x_1} + \frac{\partial u_1}{\partial x_2} \\
 \gamma_{13} &= \frac{\partial u_3}{\partial x_1} + \frac{\partial u_1}{\partial x_3} \\
 \gamma_{23} &= \frac{\partial u_3}{\partial x_2} + \frac{\partial u_2}{\partial x_3}
 \end{aligned} \tag{F.2}$$

Stress [65]

The state of internal force at a point of the body is defined by nine components of stress:

$$\begin{array}{ccc}
 \sigma_{11} & \tau_{12} & \tau_{13} \\
 \tau_{21} & \sigma_{22} & \tau_{23} \\
 \tau_{31} & \tau_{32} & \sigma_{33}
 \end{array} \tag{F.3}$$

which should satisfy the equations of equilibrium:

$$\begin{aligned}
\frac{\partial \sigma_{11}}{\partial x_1} + \frac{\partial \tau_{12}}{\partial x_2} + \frac{\partial \tau_{13}}{\partial x_3} + U_1 &= 0 \\
\frac{\partial \tau_{21}}{\partial x_1} + \frac{\partial \sigma_{22}}{\partial x_2} + \frac{\partial \tau_{23}}{\partial x_3} + U_2 &= 0 \\
\frac{\partial \tau_{31}}{\partial x_1} + \frac{\partial \tau_{32}}{\partial x_2} + \frac{\partial \sigma_{33}}{\partial x_3} + U_3 &= 0
\end{aligned} \tag{F.4}$$

and

$$\begin{aligned}
\tau_{12} &= \tau_{21} \\
\tau_{13} &= \tau_{31} \\
\tau_{23} &= \tau_{32}
\end{aligned} \tag{F.5}$$

where U_1 , U_2 and U_3 are components of the body forces per unit volume. By eliminating τ_{21} , τ_{31} and τ_{32} by the use of (F.5), then (F.4) becomes:

$$\begin{aligned}
\frac{\partial \sigma_{11}}{\partial x_1} + \frac{\partial \tau_{12}}{\partial x_2} + \frac{\partial \tau_{13}}{\partial x_3} + U_1 &= 0 \\
\frac{\partial \tau_{12}}{\partial x_1} + \frac{\partial \sigma_{22}}{\partial x_2} + \frac{\partial \tau_{23}}{\partial x_3} + U_2 &= 0 \\
\frac{\partial \tau_{13}}{\partial x_1} + \frac{\partial \tau_{23}}{\partial x_2} + \frac{\partial \sigma_{33}}{\partial x_3} + U_3 &= 0
\end{aligned} \tag{F.6}$$

Strain [65]

The state of strain at a point of the body is defined by six components of strain, namely:

$$\epsilon_{11}, \epsilon_{22}, \epsilon_{33}, \gamma_{12}, \gamma_{13} \text{ and } \gamma_{23}. \tag{F.7}$$

Sobolev spaces of functions [38]

Sobolev spaces of functions are defined as follows:

$$H^k = H^k(\Omega) = \{w | w \in L_2; w_{,x} \in L_2; \dots; w_{,x\dots x} \in L_2\} \tag{F.8}$$

where

$$L_2 = L_2(\Omega) = \{w | \int_0^1 w^2 dx < \infty\} \tag{F.9}$$

In words, the Sobolev space of degree k , denoted H^k , consists of functions that possess square-integrable generalized derivatives through order k . A square-integrable function is called an L_2 -function, by virtue of (F.9). From (F.8), it can be seen that:

$$H^0 = L_2 \tag{F.10}$$

and that

$$H^{k+1} \subset H^k \tag{F.11}$$

Variational form [64]

In the variational solution of differential equations, the differential equation is put into an equivalent variational form, and then the approximate solution is assumed to be a combination, $\sum c_j \phi_j$, of given approximation functions ϕ_j . The parameters c_j are determined from the variational form. The variational methods suffer from the disadvantage that the approximation functions for problems with arbitrary domains are difficult to construct.