# DEVELOPMENT OF A METHODOLOGY FOR CALCULATING STRESSES IN TRACK COMPONENTS

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Submitted in fulfilment of part of the requirements, for the degree of

**MASTER OF ENGINEERING (MECHANICAL)** 

in the

**FACULTY OF ENGINEERING** 

UNIVERSITY OF PRETORIA

October 2004



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### **SUMMARY**

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An existing analytical model, in use by Spoornet for the past two decades for calculating rail stresses on railway track, was revisited and improved. The model provided engineers with an easy-to-use program for evaluating track capacity and authorizing heavier loads on track. The model was modified to calculate rail and track component stresses more accurately. These modifications include the incorporation of current best practices and presentation of guidelines for the engineer on how to determine some input parameters which are normally difficult to obtain.

Firstly it was determined which input parameters the model was the most sensitive to. Thereafter it was determined whether or not the correct information would generally be readily available for those sensitive parameters. The most sensitive parameters were further investigated and test results, as well as best practice analytical methods, were used to establish nominal input values and guidelines for determining such values.

This research was necessary to establish whether or not the currently used analytical model still provided railway engineers with a useful tool and whether or not more modern and popular tools could validate or replace it.

After some modifications to the analytical model, it was proved that it provides engineers with a suitably accurate tool for calculating rail and track component stresses, without the need to build time-consuming models of the track under investigation. It showed that the model, after some modifications, is current with calculational methods in recent publications and provides an immediate answer to "what-if" questions without the need to run lengthy analyses.







### **ACKNOWLEDGEMENTS**

I would like to thank and acknowledge contributions by the following persons:

- All my former Spoornet colleagues for their invaluable assistance and support during this research.
- Mr Hannes Maree for supporting this project and for providing valuable information and guidelines.
- Mr Mike Tomas for sharing his expertise and unlimited supply of test data.
- Professor Nico Theron for his support, enthusiasm and guidance.
- Dr Robert Fröhling for his consistent vision, technical contributions and for his driving force without which this project may not have been completed!
- Spoornet and Land Mobility Technologies (Pty) Ltd for making this research possible.
- I wish to thank my wife for all the hours I spent in seclusion and for her everlasting understanding, support and love, and my daughter, Andrea, whose birthdate marked a D-date for the delivery of this dissertation and forced many a late night's work!

Lastly, I want to echo my wife and thank my Lord and Saviour, in heaven, here and everywhere, for the opportunity to have an occupation that keeps me busy (Ecclesiastes 3:10).







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### **NOMENCLATURE**

A	Section area
$A_e$	Effective area
$A_{rs}$	Baseplate area
a	Sleeper spacing
$a_0$	Coefficient of locomotive maintenance
$a_1$	Railhead depth
$a_2$	Top of rail to top of flange
$\alpha$	Coefficient of thermal expansion
$b_1$	Width of railhead
$b_2$	Width of rail flange
β	Inverse of the characteristic length
$b_0$	Coefficient of track maintenance
C	Foundation modulus
$C_I$	Torsion stiffness
C	Actual super elevation
D	Wheel diameter [m]
d	Section height
$\delta$	Running top condition
E	Elasticity modulus
e	Eccentricity of vertical forces
$F_0$	Pretension force of fastening system
f	Flange thickness
fL	Factor of static load
g	Gravitational acceleration
h	Head NA to flange NA
h	Track cant
$h_1$	Distance from head NA to section rotation point
$h_2$	Distance from flange NA to section rotation point
$\eta$	Influence factor of adjoining wheels







### **NOMENCLATURE**

Ι Second moment of area

 $I_{vv}$ Second moment of area yy

Second moment of area zz  $I_{zz}$ 

Second moment of area yy head  $I_{vv}$  Head

Second moment of area zz head  $I_1$  or  $I_{zh}$ 

 $I_2$  or  $I_{zf}$ Second moment of area zz flange

Dynamic factor  $\phi$ 

k Equivalent spring stiffness (MN/m)

Characteristic value ĸ

λ Characteristic length

Lateral load L

NA Neutral Axis

Position of lateral force below rail top n

P Static wheel load

Vehicle centre of gravity height  $p_c$ 

Dynamic wheel load Q

Reaction force per unit area q'

Curve radius R

R'Sleeper reaction force

Track width (gauge) S

Stress  $\sigma$ 

Mean contact pressure between rail and sleeper  $\sigma_{rs}$ 

TTorsion

 $\Delta T$ Temperature change

Web thickness

Statistical confidence level

 $\Delta T$ Max temp change

UTS Ultimate Tensile Strength

UTrack modulus







### **NOMENCLATURE**

υ	Poisson
V	Speed (km/h)
ν	Type of sleeper
x	Distance from wheel
$x_I$	Distance from wheel in front
$x_2$	Distance from wheel behind
$x_3$	Distance from third wheel
Y	Lateral load
yk	Top to section NA
Z	Deflection of beam





