

Dynamic Cyclic Bending, Kinetic to Strain Energy, Deceleration Systems.

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and the effect of varying boundary conditions on the parallel profile strip method. The overall objective was to determine the effect of varying boundary conditions, namely, the initial condition at the inlet, the outlet condition, the boundary condition at the top and bottom boundaries, on the parallel profile strip method and the performance characteristics of the parallel profile strip method under different boundary conditions. The effect of varying boundary conditions on the parallel profile strip method is discussed.

The parallel profile strip method is a numerical technique used to solve two-dimensional problems. The parallel profile strip method is a finite difference technique that uses a grid of points to represent the domain of the problem. The method is based on the principle of superposition, which states that the solution to a problem can be obtained by summing the solutions to a series of smaller, simpler problems. The parallel profile strip method is particularly well suited for solving problems involving complex boundary conditions, such as those found in the study of flow in porous media or in the study of heat transfer in composite materials.

Abstract

During the period of 2001 to 2003 an investigation has been completed at the University of Pretoria, which explored the application of a cyclic plastic bending deceleration system in a dynamic environment. The system consisted of a user-defined number of inline guide rollers, with a metal element threaded in an over and under arrangement. This metal element, when drawn through the rollers, was subjected to bending and reverse bending as the material moved from roller to roller. This action allowed for kinetic energy to be absorbed by means of the kinetic to strain energy transformation mechanism. In the past, this concept and the performance prediction thereof had been proved reliable under quasi-static conditions however, the performance prediction under dynamic conditions remained uncharted.

The study performed at the University of Pretoria produced a dynamic performance prediction model. The various predictions were compared to scale experimental simulations, as well as dynamic finite element analyses. The comparison between the predictions, (mathematical prediction, experimental model results and the finite element analyses) proved to be extremely accurate and consistent. A full-scale evaluation would be the next remaining step to evaluate the system's suitability for future application in industry.

Opsomming

Gedurende die periode van 2001 tot 2003 was ‘n studie afgehandel by die Universiteit van Pretoria, ten einde die toepassing van sikliese plastiese buigings vertraginge sisteme, onder dinamiese toestande. Die sisteme behels n gebruiker-gespesifieerde hoeveelheid staal rollers wat inlyn langs mekaar gerangskik is. ‘n Staal strook word tussen die rollers deur gevleg. Wanneer die strook aan die een punt tussen die rollers deur gesleep word, buig die strook herhaaldelik om die spoor tussen die rollers deur te volg. Hierdie buigings proses absorbeer energie deur middle van omskakeling van kinetiese na vervormings energie. In die verlede is n model ontwikkel om briekkrag lewering onder kwasi-satiese toestande betroubaar te voorspel. ‘n Dinamiese lewerings model is nog nie ontwikkel nie.

Tydens die Universiteit van Pretoria se ondersoek is n dinamiese model ontwikkel en wiskundig geprogrammeer, wat dan die briekkrag lewering van die sisteem onder dinamiese toestande voorspel. Hierdie wiskundige voorspellings is dan vergelyk met ‘n afgeskaalde eksperimentele model wat gebou is by die Universiteit van Pretoria, asook ‘n dinamiese eindige element analise van die model. Die vergelyking van al drie resultate, (wiskundige model, eksperimentele opstelling en die eindige element analise) was baie goed en herhaalbaar. Die volgende logiese ontwikkelings stadium in voorbereiding vir industriële toepassing van hierdie sisteme, sal dus volskalse toetse wees.

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List of symbols

a	Acceleration. (m/s^2)
D	Coefficient in stress strain relation. (Mild steel characteristic)
F	Force applied by deceleration system (N)
FEM	Finite Element Method
FEML	Finite Element Model
FEA	Finite Element Analysis
Gs	Gravitational acceleration. (9.8m/s^2)
M	Mass of cage. (kg)
Msec	Millisecond. (0.001sec)
n	Number of rollers in roller assembly.
p	Exponent in stress strain relation.
R	Roller radius (m)
t	Strip thickness (m)
w	Roller width (m)
σ_{yd}	Dynamic yield stress approximation. (Mpa)
σ_{yo}	Static yield stress of material. (Mpa)
$\dot{\varepsilon}'$	Strain rate at the point of plastic deformation. (/s)
ε_x	Strain at distance y from neutral plane.
y	Distance from neutral plane. (m)
ρ	Curvature radius of neutral plane. (m)

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